Delay-Compensated Distributed PDE Control of Traffic with Connected/Automated Vehicles

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Abstract—We develop an input delay-compensating design for stabilization of an Aw-Rascle-Zhang (ARZ) traffic model in congested regime which is governed by a 2×2 first-order hyperbolic nonlinear PDE. The traffic flow consists of both adaptive cruise control-equipped (ACC-equipped) and manually-driven vehicles. The control input is the time gap of ACC-equipped and connected vehicles, which is subject to delays resulting from communication lag. For the linearized system, a novel three-branch bakestepping transformation with explicit kernel functions is introduced to compensate the input delay. The transformation is proved to be bounded, continuous and invertible, with explicit inverse transformation derived. Based on the transformation, we obtain the explicit predictor-feedback controller. We prove exponential stability of the closed-loop system with the delay compensator in L_2 norm. The performance improvement of the closed-loop system under the proposed controller is illustrated in simulation.

Index Terms—Delayed distributed input, PDE backstepping, Traffic flow, Predictor-feedback, First-order hyperbolic PDE, Adaptive cruise control (ACC)

I. INTRODUCTION

RAFFIC congestion has become a severe worldwide social issue. Stop-and-go traffic is a common phenomenon in congested highway traffic [30], which results from a small perturbation, such as a delay in a driver's response, propagating backward in traffic flow [8], [20]. The stop-and-go oscillation in traffic flow leads to poorer driving experience, higher fuel consumption and a high accident risk. One promising way to reduce the oscillation in the congested regime is to develop control design tools that exploit the capabilities of automated and connected vehicles, such as manipulation of the time gap setting of ACC-equipped and connected vehicles [4], [12].

PDE-based models have established a realistic description of the traffic dynamics [3], [10], [11], [17], [29] by capturing the temporal and spatial dynamics of the traffic density and the traffic speed along the considered highway stretch. Boundary control [16], [33]–[35], [37], [38], and in-domain manipulation [9], [31] are both developed to stabilize the traffic flow. Traffic state estimations are also considered in different situations

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[35], [5], [13]. Due to the information transmission or (and) reaction time of drivers [7], there are usually time delays in the traffic flow control process, see, e.g., [15] for delays in feedback. Therefore, it is necessary to study the traffic flow control involving delays. However, few papers on traffic control consider delay compensation.

We consider an Aw-Rascle-Zhang (ARZ) traffic model in congested regime comprising manually-driven vehicles and ACC vehicles, which are subject to input delays. The model is composed of 2×2 first-order hyperbolic PDEs whose states are traffic density and traffic speed. In order to eliminate the stop-and-go waves, a distributed actuation using the time-gap of the ACC vehicles is employed for control. If the actuator delay is present and sufficiently large, the state feedback control proposed in [4] may become destabilizing.

Relevant advances have been achieved towards the controlling of diffusion-driven distributed parameter systems with delays. Early predictor-based boundary controller is developed using the PDE backstepping method in [18], which stabilizes an unstable reaction-diffusion PDE with arbitrarily long input delay. The aforementioned method has been extended to a 3-D formation control problem to compensate for the effect of potential input delays [25]. For state delays in an unstable reaction-diffusion PDE, a boundary feedback has been developed using backstepping in [14]. Recently, a delaycompensator was designed for an unstable reaction-diffusion PDE via distributed actuation in [24]. Alternatively, series control design approaches based on Lyapunov-Krasovskii functions have been proposed in [27] and [28], respectively. A boundary feedback to compensate a constant input delay for an unstable reaction-diffusion PDE has been developed in [22], using spectral reduced-order models which approximate the infinite-dimensional system by a finite-dimensional one. A similar method is used for in-domain stabilization of reactiondiffusion PDEs with time-and spatially-varying delays in [19].

Most studies on delay-compensator design are for parabolic PDEs. There are fewer results on hyperbolic PDEs with delays. An example of first-order hyperbolic PDE is given in [26], where a backstepping boundary control is designed for compensating the input delay. State delay and measurement delay is addressed in [23]. In the context of robustness analysis, a delay-robust boundary feedback has been proposed for a 2×2 linear hyperbolic PDEs in [2]. Both in [6] and [32], the authors introduce an equivalent delay system representation to first-order hyperbolic PDEs, which transforms the coupled PDE-ODE systems to ODE systems with input delays.

The overall challenge addressed in this work is the design of an in-domain delay-compensator for a 2×2 hyperbolic PDE by

employing the PDE backstepping method while dealing with a dynamic boundary condition which results in a 2×2 hyperbolic PDE-ODE cascade system. The usual Volterra integral transformation cannot be applied directly to control design because the resulting kernel equation is unsolvable. Therefore, we propose a three-branch affine Volterra transformation which contains the state of the ODE, namely, the traffic speed at the outlet boundary. The transformation with explicit kernel functions has a different form in each of the three intervals. Although there are three intervals, the transformation is proved to be continuous in its domain. Further, we derive the explicit inverse transformation which is bounded and continuous too. Based on the transformation, we obtain an explicit delaycompensator, composed of the feedback of the states and the historical actuator state. The compensator stabilizes the traffic flow with input delay via manipulation of the time gap of the ACC-equipped vehicles in-domain. We prove the closedloop system is L_2 exponentially stable, by establishing the L_2 stability of the target system and the norm equivalence between the target system and the original system based on the fact that the transformation is invertible.

The structure of the paper is as follows. In Section II, we introduce the model. In Section III, we design the delay-compensator using the backstepping method. Section IV presents the proof of the L_2 norm exponential stability of the close-loop system. The effectiveness of the proposed delay-compensated controller is illustrated with numerical simulations in Section V. The conclusion is given in Section VI.

Notation: Throughout the paper, we adopt the following notation to define L_2 -norm for $f(\cdot) \in L_2(0,L), g(\cdot,\cdot) \in L_2((0,L)\times(0,D))$:

$$||f||_{L_2}^2 = \int_0^L |f(x)|^2 dx, \quad ||g||_{L_2}^2 = \int_0^L \int_0^D |g(x,s)|^2 ds dx.$$

II. MODEL DESCRIPTION

A. ARZ Traffic model with mixed vehicles

We consider the ARZ traffic model of highway introduced in [4] but an input delay which acts on adaptive cruise controlequipped (ACC-equipped) vehicles is addressed. The state variables of the model are the traffic density $\ddot{\rho}(x,t)$ and the traffic speed $\breve{v}(x,t)$, both defined in domain $(x,t) \in$ $[0,L] \times \mathbb{R}^+$ where t is time, x is the spatial variable denoting the position on the concerned highway. Constant L > 0denotes the length of the concerned highway stretch. Define $\breve{v}(x,t) \in (0,v_{\rm f}]$ with $v_{\rm f}$ being free-flow speed. We consider a mixed traffic, consisting of both manual and ACC-equipped vehicles with the percentage of ACC-equipped vehicles with respect to total vehicles being α . Let $h_{\rm acc}(x,t)$ denote the time-gap of the ACC-equipped vehicle at x from its leading vehicle, which is the control input because a vehicle with ACC can automatically adjust its speed to maintain a desired distance (or, say, a time-gap) from vehicles ahead. Due to the lag of information transmission from the control center to each individual ACC vehicle, there often exists input delay.

Expressed with equations, the traffic flow control system we consider is:

$$\begin{split}
\check{\rho}_{t}(x,t) &= -\check{\rho}_{x}(x,t)\check{v}(x,t) - \check{\rho}(x,t)\check{v}_{x}(x,t), \\
\check{v}_{t}(x,t) &= -\check{\rho}(x,t)\frac{\partial V_{\mathrm{mix}}(\check{\rho}(x,t),\check{h}_{\mathrm{acc}}(x,t-D))}{\partial \check{\rho}}\check{v}_{x}(x,t) \\
&- \check{v}(x,t)\check{v}_{x}(x,t) \\
&+ \frac{V_{\mathrm{mix}}(\check{\rho}(x,t),\check{h}_{\mathrm{acc}}(x,t-D)) - \check{v}(x,t)}{\tau_{\mathrm{mix}}(\alpha)},
\end{split} (2)$$

$$\tilde{\rho}(0,t) = q_{\rm in}/\tilde{v}(0,t), \tag{3}$$

$$\ddot{v}_t(L,t) = \frac{V_{\text{mix}}(\breve{\rho}(L,t), \breve{h}_{\text{acc}}(L,t-D)) - \breve{v}(L,t)}{\tau_{\text{mix}}(\alpha)},
 \tag{4}$$

where D is the delay on the domain-wide actuated time gap input, and

$$\tau_{\text{mix}}(\alpha) = \frac{1}{\frac{\alpha}{\tau_{\text{acc}}} + \frac{1 - \alpha}{\tau_{\text{m}}}}, \ 0 \le \alpha \le 1$$
 (5)

is time constant for a mixture traffic which depends on both time constant $\tau_{\rm acc}$ of ACC vehicles and time constant $\tau_{\rm m}$ of manual vehicles. $\tau_{\rm mix}$ is also a function of α , the percentage

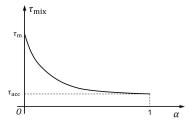


Fig. 1: $\tau_{\rm mix}$ varies with α .

of ACC vehicles with respect to total vehicles. Fig. 1 shows the relations of these two parameters. The equilibrium speed profile of the mixed flow $V_{\rm mix}$ is expressed as

$$V_{\rm mix}(\breve{\rho},\breve{h}_{\rm acc}) = \frac{1}{\breve{h}_{\rm mix}(\breve{h}_{\rm acc})} \left(\frac{1}{\breve{\rho}} - l\right),\tag{6}$$

and the mixed time gap is defined as

$$\check{h}_{\text{mix}}(\check{h}_{\text{acc}}) = \frac{\alpha + (1 - \alpha)\frac{\tau_{\text{acc}}}{\tau_{\text{m}}}}{\alpha + (1 - \alpha)\frac{\tau_{\text{acc}}}{\tau_{\text{m}}}\frac{\check{h}_{\text{acc}}}{\check{h}_{\text{-}}}\check{h}_{\text{-}}}\check{h}_{\text{acc}}.$$
(7)

In the above model, l>0 denotes the average effective vehicle length, $q_{\rm in}>0$ is a constant external inflow, and $\check{h}_{\rm m}>0$ is the time gap of manual vehicles.

Equation (1) means that the traffic flow observe the mass conservation law [16]. Equation (2) is a momentum equation inspired by the speed dynamics of ARZ model [36] for ACC-equipped and manual mixed flow, where $\alpha \in [0,1]$ is the percentage of ACC vehicles with respect to total vehicles. In (2), $V_{\rm mix}(\check{\rho},\check{h}_{\rm acc}) = Q(\check{\rho},\check{h}_{\rm acc})/\check{\rho}$ is the equilibrium speed profile of a mixed flow of ACC vehicles and manual vehicles, where $Q(\check{\rho},\check{h}_{\rm acc})$ is the traffic flow given by the fundamental diagram shown in Fig. 2. Define $\check{\rho}_c$ as the lowest density value of the mixed time gap $\check{h}_{\rm mix}$, for which the traffic is congested. Let $\check{h}_{\rm min}$ and $\check{h}_{\rm max}$ be the minimum and maximum possible time gap, namely, $\check{h}_{\rm min} \leq \min\{\check{h}_{\rm acc},\check{h}_{\rm m}\}$ and

 $\check{h}_{\max} \geq \max\{\check{h}_{\mathrm{acc}}, \check{h}_{\mathrm{m}}\}$. Define $\check{\rho}_{\min}$ and $\check{\rho}_{\max}$ as the lowest density values of congested traffic that correspond to minimum and maximum possible time gaps \check{h}_{\min} and \check{h}_{\max} , respectively. From Fig. 2. we find $Q_{\check{h}_{\min}}(\check{\rho}_{\min}) = v_{\mathrm{f}}\check{\rho}_{\min}$ is the maximal flow at given time gap $\check{h}_{\min} \in [\check{h}_{\min}, \check{h}_{\max}]$. If $\check{\rho} \geq \check{\rho}_{\min}$, implying that the traffic is in congested state, we have

$$Q_{\check{h}_{\text{mix}}}(\check{\rho}) = (1 - l\check{\rho}) \frac{v_{\text{f}}}{1/\check{\rho}_{\text{c}} - l}.$$
 (8)

Combined with (6), we get

$$Q_{\check{h}_{\text{mix}}}(\check{\rho}, \check{h}_{\text{acc}}) = \check{\rho}V_{\text{mix}} = (1 - l\check{\rho})\frac{1}{\check{h}_{\text{mix}}(\check{h}_{\text{acc}})}, \qquad (9)$$

which gives

$$\check{h}_{\text{mix}}(\check{h}_{\text{acc}}) = \frac{1/\check{\rho}_{\text{c}} - l}{v_{\text{f}}}.$$
(10)

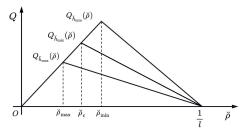


Fig. 2: Fundamental diagrams for $\check{h}_{\text{mix}} \in [\check{h}_{\text{min}}, \check{h}_{\text{max}}]$.

In other words, for each input $\check{h}_{\rm mix}$, there is a corresponding $\check{\rho}_{\rm c}$, such as $\check{\rho}_{\rm min} = \frac{1}{\check{h}_{\rm min}v_{\rm f}+l}$, which guarantees $0 < V_{\rm mix}(\check{\rho},\check{h}_{\rm acc}) \leq v_{\rm f}$. Fig. 2 shows that possible flow $Q_{\check{h}_{\rm mix}}(\check{\rho})$ for every $\check{h}_{\rm mix} \in [\check{h}_{\rm min},\ \check{h}_{\rm max}]$ lies between $Q_{\check{h}_{\rm max}}(\check{\rho})$ and $Q_{\check{h}_{\rm min}}(\check{\rho})$. In congested regime, we define the feasible set of the state and input variables: $\Phi = \{(\check{v},\check{\rho},\ \check{h}_{\rm acc}) \in \mathbb{R}^3,\ 0 \leq \check{v} \leq v_{\rm f},\ \check{\rho}_{\rm min} \leq \check{\rho} \leq 1/l,\ \check{h}_{\rm min} \leq \check{h}_{\rm acc} \leq \check{h}_{\rm max}\}$, for all $\alpha \in [0,1]$.

Using an analysis method similar to the one employed in [36], one can find that system (1)-(4) is anisotropic.

B. Linearization of the ARZ model

Consider the same equilibria of system (1)-(4) as in [4], dictated by a constant inflow $q_{\rm in}$ and a constant, steady-state time gap $\bar{h}_{\rm acc}$ for ACC vehicles, which results in the following steady-state traffic speed and density:

$$\bar{v} = \frac{l}{(1/q_{\rm in}) - \bar{h}_{\rm mix}}, \quad \bar{\rho} = \frac{1}{l + \bar{h}_{\rm mix}\bar{v}},$$
 (11)

with mixed time gap

$$\bar{h}_{\text{mix}} = \frac{\alpha + (1 - \alpha) \frac{\tau_{\text{acc}}}{\tau_{\text{m}}}}{\alpha + (1 - \alpha) \frac{\tau_{\text{acc}}}{\tau_{\text{m}}} \bar{h}_{\text{acc}}} \bar{h}_{\text{acc}}.$$
 (12)

We define the error variables

$$\begin{split} \rho(x,t) &= \breve{\rho}(x,t) - \bar{\rho}, \\ v(x,t) &= \breve{v}(x,t) - \bar{v}, \\ h_{\rm acc}(x,t) &= \breve{h}_{\rm acc}(x,t) - \bar{h}_{\rm acc}. \end{split}$$

Linearizing (1)-(4) around the equilibrium (11) and (12), we get

$$\rho_{t}(x,t) = -\bar{v}\rho_{x}(x,t) - \bar{\rho}v_{x}(x,t), \tag{13}$$

$$v_{t}(x,t) = \frac{l}{\bar{h}_{\text{mix}}}v_{x}(x,t) - \frac{1}{\bar{\rho}^{2}\tau_{\text{mix}}\bar{h}_{\text{mix}}}\rho(x,t) - \frac{1}{\tau_{\text{mix}}}v(x,t) - \frac{\alpha(1-l\bar{\rho})}{\tau_{\text{acc}}\bar{h}_{\text{acc}}^{2}\bar{\rho}}h_{\text{acc}}(x,t-D), \tag{14}$$

$$\rho(0,t) = -\frac{\bar{\rho}}{\bar{v}}v(0,t),\tag{15}$$

$$v_t(L,t) = -\frac{1}{\bar{\rho}^2 \tau_{\text{mix}} \bar{h}_{\text{mix}}} \rho(L,t) - \frac{1}{\tau_{\text{mix}}} v(L,t) - \frac{\alpha(1-l\bar{\rho})}{\tau_{\text{acc}} \bar{h}_{\text{acc}}^2 \rho} h_{\text{acc}}(L,t-D).$$
(16)

Introducing a change of variable

$$z(x,t) = e^{\frac{x}{\bar{v}\tau_{\text{mix}}}} \left(\rho(x,t) + \bar{h}_{\text{mix}}\bar{\rho}^2 v(x,t)\right), \tag{17}$$

and denoting the input by $u(x,t)=h_{\rm acc}(x,t)$, we obtain a 2×2 first-order hyperbolic linear PDE system in a diagonal form

$$z_t(x,t) = -c_1 z_x(x,t) - e^{c_2 x} c_3 u(x,t-D),$$

$$v_t(x,t) = c_4 v_x(x,t) - c_5 e^{-c_2 x} z(x,t) - c_6 u(x,t-D),$$
(19)

$$z(0,t) = -c_7 v(0,t), (20)$$

$$v_t(L,t) = -c_5 e^{-c_2 L} z(L,t) - c_6 u(L,t-D),$$
 (21)

$$z(x,0) = z_0(x), \quad v(x,0) = v_0(x),$$
 (22)

$$u(x, s - D) = \vartheta_0(x, s), \quad s \in [0, D],$$
 (23)

where $c_1=\bar{v},\,c_2=\frac{1}{\tau_{\mathrm{mix}}\bar{v}},\,c_3=\frac{\alpha\bar{h}_{\mathrm{mix}}\bar{\rho}^2}{\tau_{\mathrm{acc}}h_{\mathrm{acc}}^2}((1/\bar{\rho})-l),\,c_4=\frac{l}{\bar{h}_{\mathrm{mix}}},\,c_5=\frac{1}{\bar{\rho}^2\tau_{\mathrm{mix}}\bar{h}_{\mathrm{mix}}},\,c_6=\frac{\alpha}{\tau_{\mathrm{acc}}h_{\mathrm{acc}}^2}((1/\bar{\rho})-l),\,\,c_7=\frac{l\bar{\rho}^2}{\bar{v}},\,\,\mathrm{and}$ one can easily find the equivalence relation of the coefficients: $c_2c_4=c_5c_7$ and $c_1c_2=\frac{c_3c_5}{c_6}$. The initial conditions is defined in (22). The initial actuator state, i.e., the control memory in [0,D], is denoted by $\vartheta_0(x,s)\in L_2([0,L]\times[0,D])$ in (23).

Before we proceed, we make the following assumption on the coefficients:

Assumption 1. Assume $(c_1 + c_4)D < L$, which gives $(c_1 + c_4)s < L$ for all $0 \le s \le D$.

Remark 1. The assumption is reasonable for the traffic application, because the length of the concerned highway stretch L is usually far greater than the other parameters, such that $(c_1+c_4)D$ (delay D times the sum of steady speed $c_1=\bar{v}$ and vehicle length l over mixed time gap $\bar{h}_{\rm mix}$, $c_4=\frac{l}{\bar{h}_{\rm mix}}$) much less than L.

Our goal is to find a control u(x,t) that exponentially stabilizes the linearized system (18)-(23) with input delay. In the next section, we present the control design.

III. PREDICTOR-FEEDBACK CONTROL DESIGN

Before we apply the PDE backstepping approach to the linearized model (18)-(23) with input delay, we first introduce

a 2D transport PDE representation of the delay on the 1Ddistributed input:

$$z_t(x,t) = -c_1 z_x(x,t) - e^{c_2 x} c_3 \psi(x,0,t), \tag{24}$$

$$v_t(x,t) = c_4 v_x(x,t) - c_5 e^{-c_2 x} z(x,t) - c_6 \psi(x,0,t),$$
 (25)

$$z(0,t) = -c_7 v(0,t), (26)$$

$$v_t(L,t) = \dot{c}_5 e^{-c_2 L} z(L,t) - c_6 \psi(L,0,t),$$
 (27)

$$\psi_t(x, s, t) = \psi_s(x, s, t), \tag{28}$$

$$\psi(x, D, t) = u(x, t), \tag{29}$$

$$\psi(x,s,0) = \theta_0(x,s). \tag{30}$$

From the last three equations, we have

$$\psi(x,s,t) = \begin{cases} u(x,t+s-D) & s+t > D\\ \vartheta(x,t+s) & s+t \le D \end{cases}$$
(31)

A. Backstepping transformation

To design a stabilizing controller for the PDE-ODE system (24)-(29) one has to understand first both its open-loop structure and its actuation structure. Physically speaking, there are three transport processes. Two of the transport processes are in 1D and one is in 2D. One 1D transport is in the xdirection at speed c_1 in (24). The other 1D transport is in the -x direction at speed c_4 in (25). The two 1D transports create a (z, v) PDE feedback loop, which may be unstable. The 2D transport is in the -s direction at unity speed (and stagnant in the x direction) in (28).

The z(x,t) term in (25) creates a potentially destabilizing feedback loop but is matched by the actuated term $\psi(x,0,t)$. Likewise, the z(L,t) term in (27) creates a potentially destabilizing feedback loop but is matched by the actuated term $\psi(L,0,t)$. These two observations motivate the choice of a target system as

$$z_t(x,t) = -c_1 z_x(x,t) + c_1 c_2 z(x,t) - c_3 e^{c_2 x} \beta(x,0,t) - \frac{kc_3}{c_6} e^{c_2 x} v(x,t),$$
(32)

$$v_t(x,t) = c_4 v_x(x,t) - c_6 \beta(x,0,t) - kv(x,t),$$
 (33)

$$z(0,t) = -c_7 v(0,t), \tag{34}$$

$$v_t(L,t) = -kv(L,t) - c_6\beta(L,0,t), \tag{35}$$

$$\beta_t(x, s, t) = \beta_s(x, s, t), \tag{36}$$

$$\beta(x, D, t) = 0, (37)$$

where k > 0 is a free parameter which can be used to set the desired rate of stability.

Remark 2. The dynamic equation (27) and (35) on the boundary are not standard boundary conditions, which implies that the hyperbolic PDE (24)-(29) and (32)-(37) are both preceded by an ODE whose state is v(L,t). Introducing an additional one-dimensional state $X(t) \in \mathbb{R}$ and $Y(t) \in \mathbb{R}$ for (27) and (35), respectively, one can rewrite (27) as:

$$\dot{X} = -c_5 e^{-c_2 L} z(L, t) - c_6 \psi(L, 0, t), \tag{38}$$

$$v(L,t) = X(t). (39)$$

and (35) as:

$$\dot{Y} = -kY(L,t) - c_6\beta(L,0,t), \tag{40}$$

$$v(L,t) = Y(t). (41)$$

Since the additional ODEs are relatively simple, we directly use the boundary values v(L,t) and $v_t(L,t)$ in the following computation for notational brevity. Introduce the following transformation

$$\beta(x,s,t) = \psi(x,s,t) + \int_0^L \gamma(x,s,y)z(y,t)dy$$
$$+ \int_0^L \eta(x,s,y)v(y,t)dy + \mathfrak{r}(x,s)v(L,t)$$
$$+ \int_0^L \int_0^s G(x,s,y,r)\psi(y,r,t)drdy, \tag{42}$$

where $\gamma(\cdot,\cdot,\cdot)$, $\eta(\cdot,\cdot,\cdot)$, G(x,s,y,r) and $\mathfrak{r}(\cdot,\cdot)$ are kernel functions defined on $\mathcal{T}_1 = \{[0,L] \times [0,D] \times [0,L]\}, \mathcal{T}_2 =$ $\{(x,s,y,r)|[0,L]\times[0,D]\times[0,L]\times[0,s]\}$ and $\mathcal{T}_3=\{[0,L]\times[0,s]\}$ [0, D], respectively. Mapping original system (24)–(30) to target system (32)-(37) by transformation (42), one can get the following equations of \mathfrak{r} :

$$\mathfrak{r}_s(x,s) = c_4 \eta(x,s,L),\tag{43}$$

$$\mathfrak{r}(x,0) = 0, (44)$$

which gives

$$\mathfrak{r}(x,s) = \int_0^s c_4 \eta(x,\theta, L) d\theta, \tag{45}$$

and equations of γ and η , respectively:

$$\gamma_s(x, s, y) - c_1 \gamma_u(x, s, y) = -c_5 e^{-c_2 y} \eta(x, s, y), \tag{46}$$

$$\eta_s(x, s, y) + c_4 \eta_u(x, s, y) = 0,$$
(47)

$$\gamma(x, s, L) = -\frac{c_5}{c_1} e^{-c_2 L} \mathfrak{r}(x, s), \tag{48}$$

$$\gamma(x, s, L) = -\frac{c_5}{c_1} e^{-c_2 L} \mathfrak{r}(x, s), \qquad (48)$$

$$\gamma(x, 0, y) = -\frac{c_5}{c_6} e^{-c_2 x} \delta(x - y), \qquad (49)$$

$$\eta(x, s, 0) = -\frac{c_1 c_7}{c_4} \gamma(x, s, 0), \qquad (50)$$

$$\eta(x,s,0) = -\frac{c_1 c_7}{c_4} \gamma(x,s,0), \tag{50}$$

$$\eta(x,0,y) = \frac{k}{c_6}\delta(x-y). \tag{51}$$

Under Assumption 1, we solve kernel $\gamma(\cdot,\cdot,\cdot)$ and $\eta(\cdot,\cdot,\cdot)$

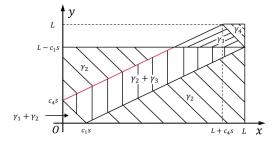


Fig. 3: The regions of kernel $\gamma(x,s,y)$.

by using the characteristic line method and the successive

approximations method (more details please see Appendix A) and get:

$$\gamma(x, s, y) = \gamma_1(x, s, y) + \gamma_2(x, s, y),$$
if $0 \le y \le c_4(s - \frac{x}{c_1});$
(52a)

$$\gamma(x, s, y) = \gamma_2(x, s, y),\tag{52b}$$

if
$$x + c_4 s < y \le L - c_1 s$$
 or $0 \le y \le x - c_1 s$;

$$\gamma(x, s, y) = \gamma_3(x, s, y), \tag{52c}$$

if
$$L - c_1 s < y \le \min\{x + c_4 s, \frac{c_1 + c_4}{c_4} L - \frac{c_1}{c_4} x - c_1 s\};$$

$$\gamma(x, s, y) = \gamma_2(x, s, y) + \gamma_3(x, s, y), \tag{52d}$$

if
$$\max\{c_4(s - \frac{x}{c_1}), x - c_1 s\} < y$$

$$\leq \min\{x + c_4 s, L - c_1 s\};$$

$$\gamma(x, s, y) = \gamma_4(x, s, y), \tag{52e}$$

if
$$\frac{c_1 + c_4}{c_4} L - \frac{c_1}{c_4} x - c_1 s \le y \le L;$$

$$\gamma(x, s, y) = 0$$
, otherwise; (52f)

with

$$\gamma_1(x, s, y) = -\frac{c_5(k + c_5 c_7)}{c_6(c_1 + c_4)} e^{-c_2(x+y)},$$
(53)

$$\gamma_2(x, s, y) = -\frac{c_5}{c_6} e^{-c_2 x} \delta(x - y - c_1 s), \tag{54}$$

$$\gamma_2(x, s, y) = -\frac{c_5}{c_6} e^{-c_2 x} \delta(x - y - c_1 s),$$

$$\gamma_3(x, s, y) = -\frac{kc_5}{c_6(c_1 + c_4)} e^{-\frac{c_1 c_2}{c_1 + c_4} x - \frac{c_2 c_4}{c_1 + c_4} (y + c_1 s)},$$
(54)

$$\gamma_4(x, s, y) = -\frac{kc_5}{c_1 c_6} e^{-c_2 L}.$$
 (56)

Fig. 3 shows all the regions that kernel $\gamma(\cdot, s, \cdot)$ takes different value under a given s, in which the red line $(y = x - c_1 s)$ displays where the pulse appears, $\gamma_2 \neq 0$. The kernel function $\gamma(x, s, y)$ with s = 4 under parameters $L = 100, c_1 = 3.1048,$ $c_2 = 0.0287, c_3 = 0.0023, c_4 = 3.5981, c_5 = 5.5671,$ $c_6 = 0.1438$, $c_7 = 0.0186$, k = 0.1, is shown in Fig. 4, where we truncate unlimited pulse of Dirac Delta function for clear displaying the kernel function. Similarly, we get

$$\eta(x, s, y) = \eta_1(x, s, y) + \eta_2(x, s, y),$$
(57a)

if
$$0 \le y \le c_4 s - \frac{c_4}{c_1} x$$
;

$$\eta(x,s,y) = \eta_2(x,s,y),\tag{57b}$$

if
$$\max\{c_4s - \frac{c_4}{c_1}x, 0\} < y \le c_4s;$$

$$\eta(x, s, y) = \eta_3(x, s, y), \tag{57c}$$

if $c_4 s < y < L$;

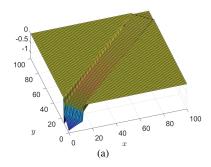
$$\eta(x, s, y) = 0, \text{ otherwise;}$$
(57d)

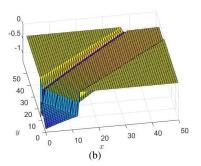
with

$$\eta_1(x, s, y) = \frac{c_1 c_5 c_7 (k + c_5 c_7)}{c_4 c_6 (c_1 + c_4)} e^{-c_2 x},$$
(58)

$$\eta_2(x, s, y) = \frac{c_1 c_5 c_7}{c_4 c_6} e^{-c_2 x} \delta(x - c_1 s + \frac{c_1}{c_4} y),$$
 (59)

$$\eta_3(x, s, y) = \frac{k}{c_6} \delta(x - y + c_4 s).$$
(60)





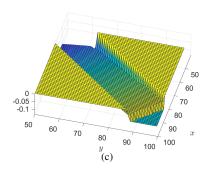


Fig. 4: (a) The kernel function $\gamma(x, s, y)$ with s = 4 and in domain $(x, y) \in ([0, 100] \times [0, 100])$, (b) the partially zoomed (a) in domain $(x, y) \in ([0, 50] \times [0, 50])$; and (c) the partially zoomed (a) in domain $(x, y) \in ([50, 100] \times [50, 100])$.

The equation of $G(\cdot,\cdot,\cdot,\cdot)$ depends on kernel $\gamma(\cdot,\cdot,\cdot)$ and $\eta(\cdot,\cdot,\cdot)$ as follows:

$$G_s(x, s, y, r) = -G_r(x, s, y, r) = 0,$$
 (61)

$$G(x, s, y, 0) = -c_3 e^{-c_2 y} \gamma(x, s, y) - c_6 \eta(x, s, y) - c_6 \delta(L - y) \mathfrak{r}(x, s),$$
(62)

which is solved

$$G(x, s, y, r) = -c_3 e^{-c_2 y} \gamma(x, s - r, y) - c_6 \eta(x, s - r, y) - c_6 \delta(L - y) \mathfrak{r}(x, s - r).$$
(63)

Substitute the kernel functions (52), (57), (45) and (63) into the transformation (42), which gives an explicit backstepping transformation as follows:

$$\beta(x,s,t) = \begin{cases} T_1[\psi(t)](x,s) + Z_1[z(t)](x,s) \\ + Y_1[v(t)](x,s), & \text{if } 0 \le x \le c_1 s, \\ T_2[\psi(t)](x,s) + Z_2[z(t)](x,s) \\ + Y_2[v(t)](x,s), & \text{if } c_1 s < x \le L - c_4 s, \\ T_3[\psi(t)](x,s) + Z_3[z(t)](x,s) \\ - \frac{k}{c_6}v(L,t), & \text{if } L - c_4 s < x \le L, \end{cases}$$

$$(64)$$

where the operators on state $\psi(\cdot,\cdot,t)$ are

$$T_{1}[\psi(t)](x,s) = \psi(x,s,t)$$

$$-\int_{0}^{\frac{x}{c_{1}}} c_{1}c_{2}e^{-c_{1}c_{2}\tau}\psi(x-c_{1}\tau,s-\tau,t)d\tau$$

$$+\int_{\frac{x}{c_{1}}}^{s} c_{5}c_{7}e^{-c_{2}x}\psi(c_{4}(\tau-\frac{x}{c_{1}}),s-\tau,t)d\tau$$

$$-\int_{0}^{s} \int_{\max\{x-c_{1}\tau,c_{4}(\tau-\frac{x}{c_{1}})\}}^{x+c_{4}\tau} \mathfrak{g}(x,y,\tau)\psi(y,s-\tau,t)dyd\tau,$$

$$+\int_{0}^{s} k\psi(x+c_{4}\tau,s-\tau,t)d\tau \qquad (65)$$

$$T_{2}[\psi(t)](x,s) = \psi(x,s,t)$$

$$-\int_{0}^{s} c_{1}c_{2}e^{-c_{1}c_{2}\tau}\psi(x-c_{1}\tau,s-\tau,t)d\tau$$

$$+\int_{0}^{s} k\psi(x+c_{4}\tau,s-\tau,t)d\tau$$

$$-\int_{0}^{s} \int_{x-c_{1}\tau}^{x+c_{4}\tau} \mathfrak{g}(x,y,\tau)\psi(y,s-\tau,t)dyd\tau, \qquad (66)$$

$$T_{3}[\psi(t)](x,s) = \psi(x,s,t)$$

$$-\int_{0}^{s} c_{1}c_{2}e^{-c_{1}c_{2}\tau}\psi(x-c_{1}\tau,s-\tau,t)r\tau$$

$$+\int_{0}^{\frac{L-x}{c_{4}}} k\psi(x+c_{4}\tau,s-\tau,t)d\tau$$

$$+\int_{0}^{s} k\psi(x+c_{4}\tau,s-\tau,t)d\tau$$

$$-\int_{0}^{s} \int_{x-c_{1}\tau}^{\min\{x+c_{4}\tau,c(\tau)\}} \mathfrak{g}(x,y,\tau)\psi(y,s-\tau,t)dyd\tau$$

$$-\int_{0}^{s} \int_{x-c_{1}\tau}^{L} kc_{2}e^{-c_{2}L}\psi(y,s-\tau,t)dyd\tau, \qquad (67)$$

with

$$\mathfrak{g}(x,y,\tau) = \frac{kc_1c_2}{c_1 + c_4} e^{-\frac{c_1c_2}{c_1 + c_4}(x - y + c_4\tau)}, \tag{68}$$

$$\mathfrak{c}(\tau) = \frac{c_1 + c_4}{c_4} L - \frac{c_1}{c_4} x - c_1\tau. \tag{69}$$

The operators on state $z(\cdot,t)$ are

$$Z_{1}[z(t)](x,s) = \int_{0}^{c_{4}(s-\frac{x}{c_{1}})} \frac{c_{5}(k+c_{5}c_{7})}{c_{6}(c_{1}+c_{4})} e^{-c_{2}(x+y)} z(y,t) dy + \int_{c_{4}(s-\frac{x}{c_{1}})}^{x+c_{4}s} \mathfrak{t}(x,s) z(y,t) dy,$$

$$(70)$$

$$Z_{2}[z(t)](x,s) = \frac{c_{5}}{c_{6}} e^{-c_{2}x} z(x - c_{1}s, t)$$

$$+ \int_{x-c_{1}s}^{x+c_{4}s} \mathfrak{t}(x,s) z(y,t) dy, \qquad (71)$$

$$Z_{3}[z(t)](x,s) = \frac{c_{5}}{c_{6}} e^{-c_{2}x} z(x - c_{1}s, t) \qquad (72)$$

$$+ \int_{\mathfrak{c}(s)}^{L} \frac{kc_{5}}{c_{1}c_{6}} e^{-c_{2}L} z(y,t) dy + \int_{x-c_{1}s}^{\mathfrak{c}(s)} \mathfrak{t}(x,s) z(y,t) dy$$

with $\mathfrak{k}(x,s)=rac{kc_5}{c_6(c_1+c_4)}\mathrm{e}^{-rac{c_1c_2}{c_1+c_4}x-rac{c_2c_4(y+c_1s)}{c_1+c_4}}$. The operators on state $v(\cdot,t)$ are

$$Y_{1}[v(t)](x,s) = -\frac{c_{5}c_{7}}{c_{6}}e^{-c_{2}x}v(c_{4}(s-\frac{x}{c_{1}}),t)$$

$$-\frac{k}{c_{6}}v(x+c_{4}s,t)$$

$$-\int_{0}^{c_{4}(s-\frac{x}{c_{1}})}\frac{c_{1}c_{5}c_{7}(k+c_{5}c_{7})}{c_{4}c_{6}(c_{1}+c_{4})}e^{-c_{2}x}v(y,t)dy, \quad (73)$$

$$Y_{2}[v(t)](x,s) = -\frac{k}{c_{6}}v(x+c_{4}s,t). \quad (74)$$

Lemma 1. If Assumption 1 holds, the transformation (64) is bounded and continuous in $x \in [0, L]$, which transforms the original system (24)-(29) into the target system (32)-(37).

The proof of Lemma 1 is given in Appendix B.

Remark 3. The transformation of the plant (24)-(29) to the target system (32)-(37) clearly has to be a 2D backstepping transformation due to the 2D nature of the ψ -system and the β -system. But it is not just the dimensionality that shall make this 2D backstepping transformation complex. The reason for its complexity is that the 1D PDE dynamics of (z,v) evolve perpendicular to the direction of the 2D transport dynamics of ψ through which backstepping is performed. The transverse nature of the (z,v)-transport relative to the β -transport, in both the downstream and upstream direction, will make the backstepping transformation $\psi \mapsto \beta$ very complex.

B. Delay-compensated control

The control is obtained by substituting s=D into transformation (64), applying the boundary conditions (29) and (37), and using the relation (31), we have if t>D,

$$u(x,t) = U_1[u](x,t) - Z_1[z(t)](x,D)$$

$$-Y_1[v(t)](x,D), \qquad (75a)$$
if $0 \le x \le c_1 D,$

$$u(x,t) = U_2[u](x,t) - Z_2[z(t)](x,D)$$

$$-Y_2[v(t)](x,D), \qquad (75b)$$
if $c_1 D < x < L - c_4 D,$

$$u(x,t)=U_3[u](x,t)-Z_3[z(t)](x,DL,\tau)d\tau$$

$$-\frac{k}{c_6}v(L,t), \qquad \qquad (75c)$$
 if $L-c_4D < x \le L,$

where

$$U_{1}[u](x,t) = \int_{t-\frac{x}{c_{1}}}^{t} c_{1}c_{2}e^{-c_{1}c_{2}(t-\tau)}u(x-c_{1}(t-\tau),\tau)d\tau$$

$$-\int_{t-D}^{t-\frac{x}{c_{1}}} c_{5}c_{7}e^{-c_{2}x}u(c_{4}(t-\tau-\frac{x}{c_{1}}),\tau)d\tau$$

$$-\int_{t-D}^{t} ku(x+c_{4}(t-\tau),\tau)d\tau$$

$$+\int_{t-D}^{t} \int_{\max\{x-c_{1}(t-\tau),c_{4}(t-\tau-\frac{x}{c_{1}})\}}^{x+c_{4}(t-\tau)} \frac{kc_{1}c_{2}}{c_{1}+c_{4}}$$

$$\times e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(x-y+c_{4}(t-\tau))}u(y,\tau)dyd\tau, \qquad (76)$$

$$U_{2}[u](x,t) = \int_{t-D}^{t} c_{1}c_{2}e^{-c_{1}c_{2}(t-\tau)}u(x-c_{1}(t-\tau),\tau)d\tau$$

$$-\int_{t-D}^{t} ku(x+c_{4}(t-\tau),\tau)d\tau$$

$$+\int_{t-D}^{t} \int_{x-c_{1}(t-\tau)}^{x+c_{4}(t-\tau)} \frac{kc_{1}c_{2}}{c_{1}+c_{4}}e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(x-y+c_{4}(t-\tau))}$$

$$\times u(y,\tau)dyd\tau, \qquad (77)$$

$$U_{3}[u](x,t) = \int_{t-D}^{t} c_{1}c_{2}e^{-c_{1}c_{2}(t-\tau)}u(x-c_{1}(t-\tau),\tau)d\tau$$

$$-\int_{t-\frac{L-x}{c_{4}}}^{t} ku(x+c_{4}(t-\tau),\tau)d\tau$$

$$+\int_{t-D}^{t} \int_{x-c_{1}(t-\tau)}^{\min\{x+c_{4}(t-\tau),\frac{c_{1}+c_{4}}{c_{4}}L-\frac{c_{1}}{c_{4}}x-c_{1}(t-\tau)\}} \frac{kc_{1}c_{2}}{c_{1}+c_{4}}$$

$$\times e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(x-y+c_{4}(t-\tau))}u(y,\tau)dyd\tau$$

$$+\int_{t-D}^{t-\frac{L-x}{c_{4}}} \int_{\frac{c_{1}+c_{4}}{c_{4}}L-\frac{c_{1}}{c_{4}}x-c_{1}(t-\tau)}^{L-c_{4}} kc_{2}e^{-c_{2}L}u(y,\tau)dyd\tau.$$

$$(78)$$

It is because of the aforementioned transverse motion of the (z,v)-transport relative to the ψ -transport that the control law in (75) is given in three distinct forms: from the inlet to c_1D , from $L-c_4D$ to the outlet, and in between. The control (75) is for the linearized and diagonalized system (24)-(29), and the control law for system (1)-(4) around the equilibrium (11) is also required, so we rewrite it as follows: in the case of t>D,

$$\begin{split} \check{h}_{\rm acc}(x,t) &= \bar{h}_{\rm acc} + U_1 [\check{h}_{\rm acc} - \bar{h}_{\rm acc}] - \mathrm{e}^{\frac{c_2}{c_1}x} Z_1 [\check{\rho} - \bar{\rho}](x,D) \\ &- \bar{h}_{\rm mix} \bar{\rho}^2 Z_1 [\check{v} - \bar{v}](x,D) - Y_1 [\check{v} - \bar{v}](x,D), \quad (79a) \\ &\text{if } 0 \leq x \leq c_1 D, \\ \check{h}_{\rm acc}(x,t) &= \bar{h}_{\rm acc} + U_2 [\check{h}_{\rm acc} - \bar{h}_{\rm acc}] - \mathrm{e}^{\frac{c_2}{c_1}x} Z_2 [\check{\rho} - \bar{\rho}](x,D) \\ &- \bar{h}_{\rm mix} \bar{\rho}^2 Z_2 [\check{v} - \bar{v}](x,D) - Y_2 [\check{v} - \bar{v}](x,D), \quad (79b) \\ &\text{if } c_1 D < x \leq L - c_4 D, \\ \check{h}_{\rm acc}(x,t) &= \bar{h}_{\rm acc} + U_3 [\check{h}_{\rm acc} - \bar{h}_{\rm acc}] - \mathrm{e}^{\frac{c_2}{c_1}x} Z_3 [\check{\rho} - \bar{\rho}](x,D) \\ &- \bar{h}_{\rm mix} \bar{\rho}^2 Z_3 [\check{v} - \bar{v}](x,D) - \frac{k}{c_6} (\check{v}(L,t) - \bar{v}) \\ &- \int_{t-D}^{t-\frac{L-x}{c_4}} k(\check{h}_{\rm acc}(L,\tau) - \bar{h}_{\rm acc}) d\tau, \quad (79c) \\ &\text{if } L - c_4 D < x \leq L, \end{split}$$

Since the transformation (64) is continuous in x, both the control (75) for the linearized error system (24)-(29) and the control (79) for the original system (1)-(4) around the equilibrium (11) are continuous in x. The control (79) is composed of the feedback of the states and the historical actuator state, which is divided into three parts upon the spatial variable x. It implies the ACC vehicles at different position of the highway will apply different time gap strategies. Due to the length of the concerned highway stretch L being far greater than other parameters (Assumption 1), one can find that the first and the last sections are much shorter than the second section. Hence, most ACC vehicles on the highway adopt the second part of control law when they enter the middle interval $c_1D < x < L - c_4D$. In order to control the traffic flow in the interval near the exit of the highway, the feedback of the flow speed $\ddot{v}(L,t)$ at the exit is also required due to the dynamic boundary condition.

IV. STABILITY ANALYSIS

In this section, we analyze the stability of the closed-loop system. First, we state the main result concerning exponentially stability.

Theorem 1. Consider the closed-loop system consisting of plant (24) -(30) with control law (75), if the initial conditions $z(\cdot,0) \in H_1[0,L], \ v(\cdot,0) \in H_1[0,L], \ \text{and} \ \psi(\cdot,\cdot,0)) \in L_2(0,L) \times H_1[0,D]$ are compatible, then the equilibrium $(z(\cdot,\cdot),v(\cdot,\cdot),\psi(\cdot,\cdot,\cdot)) \equiv 0$ is exponentially stable in the L_2 sense, i.e., there exist positive constants ϑ and M such that the following holds for all t>0,

$$||z||_{L_{2}}^{2} + ||v||_{L_{2}}^{2} + |v(L,t)|^{2} + ||\psi||_{L_{2}}^{2} + ||\psi(L,\cdot,t)||_{L_{2}}^{2}$$

$$\leq Me^{-\vartheta t} \left(||z(\cdot,0)||_{L_{2}}^{2} + ||v(\cdot,0)||^{2} + ||v(L,0)||_{L_{2}}^{2} + ||\psi(\cdot,\cdot,0)||_{L_{2}}^{2} + ||\psi(L,\cdot,0)||_{L_{2}}^{2} \right). \tag{80}$$

The proof of the theorem consists of two steps. The first step is to prove the exponential stability of the target system in L_2 sense, and the second step is to prove the transformation is invertible by obtaining the explicit inverse transformation, which will establish the norm equivalence between the original system and the target system.

A. The stability of the target system

Before proceeding, we first define two Lyapunov functions for the original system (24)-(29) and the target system (32)-(37), respectively

$$V_{1}(t) = \|z\|_{L_{2}}^{2} + \|v\|_{L_{2}}^{2} + |v(L, t)|^{2} + \|\psi\|_{L_{2}}^{2} + \|\psi(L, \cdot, t)\|_{L_{2}}^{2},$$

$$(81)$$

$$V_{2}(t) = \|z\|_{L_{2}}^{2} + \|v\|_{L_{2}}^{2} + |v(L, t)|^{2} + \|\beta\|_{L_{2}}^{2} + \|\beta(L, \cdot, t)\|_{L_{2}}^{2}.$$

$$(82)$$

Lemma 2. Consider system (32)-(37). If the initial conditions $z(\cdot,0) \in H_1[0,L], \ v(\cdot,0) \in H_1[0,L], \ \text{and} \ \beta(\cdot,\cdot,0)) \in L_2(0,L) \times H_1[0,D]$ are compatibles, then the equilibrium $(z(\cdot,\cdot),v(\cdot,\cdot),\beta(\cdot,\cdot,\cdot)) \equiv 0$ of (32)-(37) is exponentially

stable in the L_2 sense, i.e., there exist positive constants θ and N such that the following holds for all t > 0,

$$\begin{split} \|z\|_{L_{2}}^{2} + \|v\|_{L_{2}}^{2} + |v(L,t)|^{2} + \|\beta\|_{L_{2}}^{2} + \|\beta(L,\cdot,t)\|_{L_{2}}^{2} \\ &\leq M \mathrm{e}^{-\theta t} (\|z(\cdot,0)\|_{L_{2}}^{2} + \|v(\cdot,0)\|_{L_{2}}^{2} + |v(L,0)|^{2} \\ &+ \|\beta(\cdot,\cdot,0)\|_{L_{2}}^{2} + \|\beta(L,\cdot,0)\|_{L_{2}}^{2}). \end{split} \tag{83}$$

Proof. First, define a Lyapunov function for target system (32)-(37)

$$V_{0}(t) = \int_{0}^{L} e^{-\sigma x} z^{2}(x, t) dx + k_{2} \int_{0}^{L} e^{\sigma x} v^{2}(x, t) dx$$
$$+ k_{3} v^{2}(L, t) + k_{4} \int_{0}^{L} \int_{0}^{D} e^{\sigma (s+x)} \beta^{2}(x, s, t) ds dx$$
$$+ k_{5} \int_{0}^{D} e^{\sigma s} \beta^{2}(L, s, t) ds, \tag{84}$$

where k_i , $\sigma > 0$, i = 2, 3, 4, 5, whose ranges will be determined later.

Differentiating (84) with respect to time, we get

$$\dot{V}_0(t) = I(t) + II(t) + III(t),$$
 (85)

where

$$I(t) = -2\int_{0}^{L} c_{1}e^{-\sigma x}z(x,t)z_{x}(x,t)dx$$

$$2\int_{0}^{L} c_{1}c_{2}e^{-\sigma x}z^{2}(x,t)dx$$

$$-2\int_{0}^{L} c_{3}e^{(c_{2}-\sigma)x}z(x,t)\beta(x,0,t)dx$$

$$-2\int_{0}^{L} \frac{kc_{3}}{c_{6}}e^{(c_{2}-\sigma)x}z(x,t)v(x,t)dx, \qquad (86)$$

$$II(t) = 2k_{2}\int_{0}^{L} c_{4}e^{\sigma x}v(x,t)v_{x}(x,t)dx$$

$$-2k_{2}\int_{0}^{L} ke^{\sigma x}v^{2}(x,t)dx$$

$$-2k_{2}\int_{0}^{L} c_{6}e^{\sigma x}v(x,t)\beta(x,0,t)dx$$

$$-k_{3}c_{6}v(L,t)\beta(L,0,t) - 2k_{3}kv^{2}(L,t), \qquad (87)$$

$$III(t) = -k_{4}\sigma\int_{0}^{L}\int_{0}^{D} e^{\sigma(x+s)}\beta^{2}(x,s,t)dx$$

$$-k_{4}\int_{0}^{L} e^{\sigma x}\beta^{2}(x,0,t)dx$$

$$-k_{5}\sigma\int_{0}^{D} e^{\sigma s}\beta^{2}(L,s,t)ds - k_{5}\beta^{2}(L,0,t). \qquad (88)$$

By using Cauchy-Schwarz Inequality, Young's Inequality and letting $E=\sup_{x\in [0,L]}\{\mathrm{e}^{c_2x}\}$, we get

$$I(t) \leq \frac{2Ekc_3}{c_6} \int_0^L e^{\sigma x} v^2 dx + c_1 c_7^2 v^2(0, t)$$

$$- \left(c_1 \sigma - 2c_1 c_2 - \frac{Ec_3}{2} - \frac{Ekc_3}{2c_6} \right) \int_0^L e^{-\sigma x} z^2 dx$$

$$+ 2Ec_3 \int_0^L e^{\sigma x} \beta^2(x, 0, t) dx, \tag{89}$$

$$II(t) \le -k_2(c_4\sigma - \frac{c_6}{2} + 2k) \int_0^L e^{\sigma x} v^2 dx - k_2 c_4 v^2(0, t)$$

$$+ 2k_2 c_6 \int_0^L e^{\sigma x} \beta^2(x, 0, t) dx + \frac{p}{2} k_3 c_6 \beta^2(L, 0, t)$$

$$- (2k_3 k - \frac{k_3 c_6}{2p} - k_2 c_4 e^{\sigma L}) v^2(L, t), \tag{90}$$

where $\int_0^L \mathrm{e}^{-\sigma x} \beta^2(x,0,t) dx \leq \int_0^L \mathrm{e}^{\sigma x} \beta^2(x,0,t) dx$ is used for getting (89) and p>0 is a free parameter of Young's Inequality. Thus, we get

$$\dot{V}_{0}(t) \leq -\left[k_{2}(c_{4}\sigma - \frac{c_{6}}{2} + 2k) - \frac{2Ekc_{3}}{c_{6}}\right] \int_{0}^{L} e^{\sigma x} v^{2} dx
-\left(c_{1}\sigma - 2c_{1}c_{2} - \frac{Ec_{3}}{2} - \frac{Ekc_{3}}{2c_{6}}\right) \int_{0}^{L} e^{-\sigma x} z^{2} dx
-k_{4}\sigma \int_{0}^{L} \int_{0}^{D} e^{\sigma(x+s)} \beta^{2}(x, s, t) dx
-\left(2k_{3}k - \frac{k_{3}c_{6}}{2p} - k_{2}c_{4}e^{\sigma L}\right) v^{2}(L, t)
-(k_{4} - 2Ec_{3} - 2k_{2}c_{6}) \int_{0}^{L} \beta^{2}(x, 0, t)
-(k_{2}c_{4} - c_{1}c_{7}^{2})v^{2}(0, t) - k_{5}\sigma \int_{0}^{D} e^{\sigma s} \beta^{2}(L, s, t) ds
-(k_{5} - \frac{p}{2}k_{3}c_{6})\beta^{2}(L, 0, t).$$
(91)

Choose the parameters as:

$$\begin{cases}
\sigma > 2c_{2} + \frac{Ec_{3}}{2c_{1}} + \frac{Ekc_{3}}{2c_{1}c_{6}}, \\
p > \frac{c_{6}}{4k}, \\
k_{2} > \max\{\frac{2Ekc_{3}}{c_{6}(c_{4}\sigma - \frac{c_{6}}{2} + 2k)}, \frac{c_{1}c_{7}^{2}}{c_{4}}\}, \\
k_{3} > \frac{k_{2}c_{4}e^{\sigma L}}{2k - \frac{c_{6}}{2p}}, \\
k_{4} > 2Ec_{3} + 2k_{2}c_{6}, \\
k_{5} > \frac{2k_{3}c_{6}}{p},
\end{cases} (92)$$

such that

$$\dot{V}_{0}(t) \leq -\left[k_{2}(c_{4}\sigma - \frac{c_{6}}{2} + 2k) - \frac{2Ekc_{3}}{c_{6}}\right] \int_{0}^{L} e^{\sigma x} v^{2} dx
-\left(c_{1}\sigma - 2c_{1}c_{2} - \frac{Ec_{3}}{2} - \frac{Ekc_{3}}{2c_{6}}\right) \int_{0}^{L} e^{-\sigma x} z^{2} dx
-k_{4}\sigma \int_{0}^{L} \int_{0}^{D} e^{\sigma(x+s)} \beta^{2}(x,s,t) dx
-\left[k_{3}(2k - \frac{c_{6}}{2p}) - k_{2}c_{4}e^{\sigma L}\right] v^{2}(L,t)
-k_{5}\sigma \int_{0}^{D} e^{\sigma s} \beta^{2}(L,s,t) ds \leq -\vartheta V(t),$$

where

$$\vartheta = \min\{k_2(c_4\sigma - \frac{c_6}{2} + 2k) - \frac{2Ekc_3}{c_6}, k_4\sigma, (c_1\sigma - 2c_1c_2 - \frac{Ec_3}{2} - \frac{Ekc_3}{2c_6}), k_3(2k - \frac{c_6}{2}) - k_2c_4e^{\sigma L}, k_5\sigma\}.$$

Therefore, we get

$$V_0(t) \le V_0(0)e^{-\vartheta t}$$
. (93)

It is obvious that $V_0(t)$ defined by (84) is equivalent to V_2 defined by (82), i.e., there exist positive constants ϱ_1 and ϱ_2 , such that $\varrho_1 V_2 \leq V_0 \leq \varrho_2 V_2$. Hence, (83) is proved.

B. The inverse transformation

Lemma 3. The transformation (64) is invertible, and whose inverse transformation is

$$\psi(x, s, t) = Q_{1}[\beta(t)](x, s) + R_{1}[v(t)](x, s),$$

$$0 \le x < c_{1}s;$$

$$\psi(x, s, t) = Q_{2}[\beta(t)](x, s) + R_{2}[v(t)](x, s)$$

$$+ \mathcal{B}[z(t)](x, s),$$

$$c_{1}s < x \le L - c_{4}s;$$

$$\psi(x, s, t) = Q_{3}[\beta(t)](x, s) + R_{3}[v(t)](x, s)$$

$$+ \mathcal{B}[z(t)](x, s),$$

$$(94c)$$

$$L - c_{4}s < x < L;$$

where

$$Q_{1}[\beta(t)](x,s) = \beta(x,s,t) \\ - \int_{x}^{s} \int_{c_{4}(\tau-\frac{x}{c_{1}})}^{x+c_{4}\tau} \frac{kc_{1}c_{2}}{c_{1}+c_{4}} e^{\frac{k(x-y-c_{1}\tau)}{c_{1}+c_{4}}} \beta(y,s-\tau,t) dy d\tau \\ - \int_{0}^{x} \int_{x-c_{1}\tau}^{x+c_{4}\tau} \frac{kc_{1}c_{2}}{c_{1}+c_{4}} e^{\frac{k(x-y-c_{1}\tau)}{c_{1}+c_{4}}} \beta(y,s-\tau,t) dy d\tau \\ - \int_{0}^{s} \int_{x-c_{1}\tau}^{x+c_{4}\tau} \frac{kc_{1}c_{2}}{c_{1}+c_{4}} e^{\frac{k(x-y-c_{1}\tau)}{c_{1}+c_{4}}} \beta(y,s-\tau,t) d\tau \\ + \int_{0}^{x} c_{1}c_{2}\beta(x-c_{1}\tau,s-\tau,t) d\tau \\ - \int_{0}^{s} ke^{-k\tau}\beta(x+c_{4}\tau,s-\tau,t) d\tau, \qquad (95) \\ Q_{2}[\beta(t)](x,s) = \beta(x,s,t) \\ - \int_{0}^{s} \int_{x-c_{1}\tau}^{x+c_{4}\tau} \frac{kc_{1}c_{2}}{c_{1}+c_{4}} e^{\frac{k(x-y-c_{1}\tau)}{c_{1}+c_{4}}} \beta(y,s-\tau,t) dy d\tau \\ + \int_{0}^{s} c_{1}c_{2}\beta(x-c_{1}\tau,s-\tau,t) d\tau \\ - \int_{0}^{s} ke^{-k\tau}\beta(x+c_{4}\tau,s-\tau,t) d\tau, \qquad (96) \\ Q_{3}[\beta(t)](x,s) = \beta(x,s,t) \\ - \int_{0}^{s} \int_{x-c_{1}\tau}^{\min\{L,x+c_{4}\tau\}} \frac{kc_{1}c_{2}}{c_{1}+c_{4}} e^{\frac{k(x-y-c_{1}\tau)}{c_{1}+c_{4}}} \\ \times \beta(y,s-\tau,t) dy d\tau \\ + \int_{0}^{s} c_{1}c_{2}\beta(x-c_{1}\tau,s-\tau,t) d\tau \\ - \int_{\frac{L-x}{c_{4}}}^{s} (c_{1}c_{2}e^{\frac{k(x-L-c_{1}\tau)}{c_{1}+c_{4}}} - c_{1}c_{2}e^{-k\tau} + ke^{-k\tau}) \\ \times \beta(L,s-\tau,t) d\tau \\ - \int_{\frac{L-x}{c_{4}}}^{s} ke^{-k\tau}\beta(x+c_{4}\tau,s-\tau,t) d\tau, \qquad (97)$$

$$R_{1}[v(t)](x,s) = \frac{c_{2}c_{4}}{c_{6}} e^{k(\frac{x}{c_{1}}-s)} v(c_{4}(s-\frac{x}{c_{1}}),t)$$

$$+ \frac{k}{c_{6}} e^{-ks} v(x+c_{4}s,t)$$

$$+ \int_{c_{4}(s-\frac{x}{c_{1}})}^{x+c_{4}s} \frac{kc_{1}c_{2}}{c_{6}(c_{1}+c_{4})} e^{\frac{k(x-y-c_{1}s)}{c_{1}+c_{4}}} v(y,t)dy, \qquad (98)$$

$$R_{2}[v(t)](x,s) = \frac{k}{c_{6}} e^{-ks} v(x+c_{4}s,t)$$

$$+ \int_{x-c_{1}s}^{x+c_{4}s} \frac{kc_{1}c_{2}}{c_{6}(c_{1}+c_{4})} e^{\frac{k(x-y-c_{1}s)}{c_{1}+c_{4}}} v(y,t)dy, \qquad (99)$$

$$R_{3}[v(t)](x,s) = \int_{x-c_{1}s}^{L} \frac{kc_{1}c_{2}}{c_{6}(c_{1}+c_{4})} e^{\frac{k(x-y-c_{1}s)}{c_{1}+c_{4}}} v(y,t)dy$$

$$+ \left(\frac{c_{1}c_{2}}{c_{6}} (e^{\frac{k(x-L-c_{1}s)}{c_{1}+c_{4}}} - e^{-ks}) + \frac{k}{c_{6}} e^{-ks} \right) v(L,t), \quad (100)$$

$$\mathcal{B}[z(t)](x,s) = -\frac{c_{5}}{c_{1}} e^{c_{2}(c_{1}s-x)} z(x-c_{1}s,t). \quad (101)$$

The inverse transformation is bounded and continuous in $x \in [0, L]$.

The proof of Lemma 3 is similar to the proof of Lemma 1, so we will omit the proof due to limited space.

Lemma 4. The Lyapunov functions V_1 defined in (81) and V_2 defined in (82) are equivalent in the sense of the L_2 norm, i.e., there exist positive constants α_1 and α_2 , such that

$$\alpha_1 V_2(t) \le V_1(t) \le \alpha_2 V_2(t).$$
 (102)

Since the transformation (64) and the inverse transformation (94) are presented in explicit form and they are bounded, the L_2 norm equivalence between the Lyapunov functions V_1 and V_2 is easily established from Theorem 1.2 [1]. Combining Lemma 2-4, we reach Theorem 1.

V. SIMULATION

In this section, we illustrate our results with an numerical example. We apply the control law (79) directly on the non-linear model (1)-(4). The parameters utilized in the simulation are shown in Table I, for which we choose the same values as those in [4].

TABLE I: PARAMETERS OF SYSTEM

L = 1000 m	l = 5 m	$q_{\rm in} = 1200 \text{ veh/h}$
$\tau_{\rm acc} = 2 \text{ s}$	$\tau_{\mathrm{m}} = 60 \mathrm{\ s}$	$\ddot{h}_{\rm m} = 1 \text{ s}$
$\bar{h}_{\rm acc} = 1.5 \text{ s}$	$ \ddot{\rho}_{\min} = 37 \text{veh/km} $	

From (12), we get the value of the mixed steady-state time gap $\bar{h}_{\rm mix}=1.35~{\rm s}$. The steady-state values for density and speed derived from (11) are $\bar{\rho}=107.36~{\rm veh/km},~\bar{v}=11.18~{\rm km/h}$. The system is discretized with time step $\Delta t=0.5~{\rm s}$ and spatial step $\Delta x=5~{\rm m}$. The initial conditions are chosen as $\check{\rho}(x,0)=10\cos(8\pi x/L)$ and $\check{v}(x,0)=q_{\rm in}/\check{\rho}(x,0),$ which imitates the stop-and-go wave in congested regime. Considering the time delay $D=4~{\rm s}$ and the coefficient $k=0.1(1/{\rm s})$ of the target system, we first investigate the numerical solution of the nonlinear system (1)-(4) with the non-delay-compensation state-feedback control proposed in [4], whose results are shown in Fig. 5 (a)-(c). It is evident

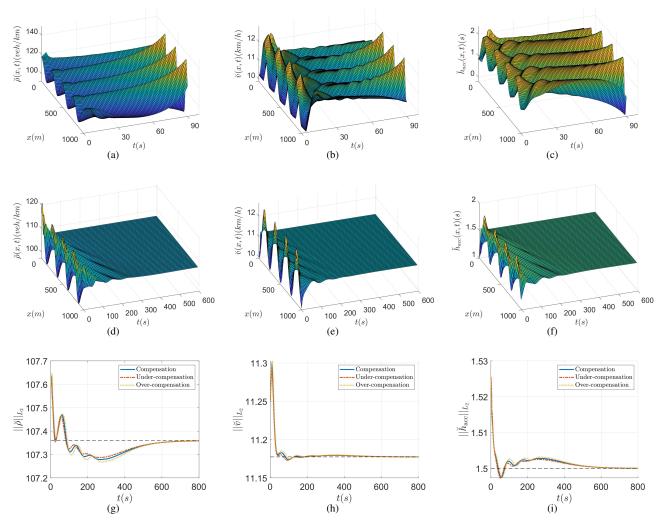


Fig. 5: The evolution of the traffic states and the actuator: (a) $\check{\rho}(x,t)$ without delay-compensation; (b) $\check{v}(x,t)$ without delay-compensation; (c) control effort $\check{h}_{\rm acc}(x,t)$ with no delay-compensation; (d) $\check{\rho}(x,t)$ with delay-compensator; (e) state $\check{v}(x,t)$ with delay-compensator; (f) control effort $\check{h}_{\rm acc}(x,t)$ with delay-compensator; (g) the L_2 norm of state $\check{\rho}(x,t)$ for delay matched and delay mismatched cases; (i) the L_2 norm of control $\check{h}_{\rm acc}(x,t)$ for delay matched and delay mismatched cases.

that the response of the system without delay-compensation exhibits an unstable and oscillatory behavior and the stopand-go wave propagates backward without being attenuated. In contrast, as it is shown in Fig. 5 (d) and (e), the traffic system (1)-(4) is stabilized under the delay-compensator and the oscillations in the speed response are suppressed first and then the oscillations in density response converges to the equilibrium as well. The control effort (79) is shown in Fig. 5 (f), which indicates the resulting values for the time gap of ACC vehicles lie within the interval [0.8, 2.2] s which is typically implemented in ACC vehicles settings, see, e.g., [21]. In order to illustrate the converging evolution of the states and the actuator more clearly, we plot the states and the control effort in L_2 norm in solid line, respectively, shown in Fig. 5 (g)-(i), where the dashed line represents the steady-state value.

To examine the robustness of the proposed delaycompensated control law, we conducted some additional simulations with mismatched delays. First, we consider the overcompensation case where the actual delay is less than the delay used in control. The numerical result is shown in Fig. 5 (g)-(i) in dotted line when the actual delay is 3s, while the delay used in control is 4s. Then, we consider the under-compensation case where the actual delay is larger than the delay used in control. The numerical result is also shown in Fig. 5 (g)-(i), where the dash-dotted line represents L_2 norm of the states $\check{\rho}$, \check{v} and the control effort, respectively, when the actual delay is 5s, while the delay used in control is 4s. From the simulation results, we find the closed-loop system remains stable in delay mismatched conditions. The performance improvement of the closed-loop system under the proposed controller (79) is also illustrated in simulation by employing three metrics, namely, total travel time (TTT), fuel consumption and comfort, where

we use the same definition as in [30]:

$$J_{\rm TTT} = \int_0^T \int_0^L \rho(x, t) dx dt, \qquad (103)$$

$$J_{\text{fuel}} = \int_0^T \int_0^L \xi(x, t) \cdot \rho(x, t) dx dt, \tag{104}$$

$$J_{\text{comfort}} = \int_0^T \int_0^L \left(a(x,t)^2 + a_t(x,t)^2 \right) \rho(x,t) dx dt, \quad (105)$$

where we also select the same functions and parameters as those in [30]: $a(x,t) = v_t(x,t) + v(x,t)v_x(x,t)$, $b_0 = 25 \cdot 10^{-3}$, $b_1 = 24.5 \cdot 10^{-6}$, $b_3 = 32.5 \cdot 10^{-9}$, $b_4 = 125 \cdot 10^{-6}$, $\xi(x,t) = \max\{0,b_0+b_1v(x,t)+b_3v(x,t)+b_4v(x,t)a(x,t)\}$ and T=300 s.

TABLE II: PERFORMANCE INDICES

Performance indices	Percentage with improvement
$J_{ m TTT}$	3.91%
$J_{ m fuel}$	3.76%
$J_{ m comfort}$	92.1%

As shown in the Table II, three indicators are improved compared to the open-loop system. Since the open-loop system is unstable, we only consider the first 300 seconds in the simulation. In particular, the driving comfort is significantly improved due to the homogenization of speed which alleviates the phenomenon of stop-and-go oscillation.

VI. CONCLUSION

In this paper, we present a control design methodology for compensation of unstable traffic flow with input delay. The delay is resulting from the time required for the transmission of the control command from the control to the ACC vehicles. Applying the PDE backstepping method, we develop an explicit feedback delay-compensator, composed of the feedback of the traffic speed, the traffic density and the historical actuator states, which is divided into three parts upon the spatial domain along the highway. The closed-loop system, under the developed compensator, was shown to be exponentially stable in the L_2 sense. Although the control design is based on a linearized system, the numerical simulation shows the effectiveness of the proposed controller on the original nonlinear system. Further research would include delay-robustness and the output feedback control based on observer design.

APPENDIX A SOLVING KERNEL EQUATIONS

Under Assumption 1, we get the integral equations from (46)–(51) by using the characteristic line method,

$$\gamma(x,s,y) = -\frac{c_5}{c_6} e^{-c_2 x} \delta(x - y - c_1 s)$$

$$-\int_y^{y+c_1 s} \frac{c_5}{c_1} e^{-c_2 \theta} \eta(x, \frac{y-\theta}{c_1} + s, \theta) d\theta, \quad (106)$$
if $y \le L - c_1 s$,
$$\gamma(x,s,y) = -\int_0^{s - \frac{L-y}{c_1}} \frac{c_4 c_5}{c_1} e^{-c_2 L} \eta(x, \theta, L) d\theta$$

$$-\int_{y}^{L} \frac{c_{5}}{c_{1}} e^{-c_{2}\theta} \eta(x, \frac{y-\theta}{c_{1}} + s, \theta) d\theta, \qquad (107)$$
if $y > L - c_{1}s$,
$$c_{1}c_{7} \qquad y \qquad (108)$$

$$\eta(x,s,y) = -\frac{c_1 c_7}{c_4} \gamma(x,s - \frac{y}{c_4}, 0), \tag{108}$$

if
$$y \leq c_4 s$$
,

$$\eta(x, s, y) = \frac{k}{c_6} \delta(x - (y - c_4 s)), \tag{109}$$

Substitute (108) and (109) into (106) and (107), which gives three-branch expression of $\gamma(x,s,y)$

$$\gamma(x, s, y) = -\int_{0}^{y+c_{1}s} \frac{kc_{5}}{c_{6}(c_{1} + c_{4})} e^{-\frac{c_{1}c_{2}}{c_{1} + c_{4}}\theta - \frac{c_{2}c_{4}}{c_{1} + c_{4}}(y+c_{1}s)}$$

$$\times \delta(x - \theta)d\theta$$

$$+ \int_{0}^{s - \frac{y}{c_{4}}} \frac{c_{1}c_{5}c_{7}}{c_{1} + c_{4}} e^{\frac{c_{2}c_{4}}{c_{1} + c_{4}}(c_{1}\theta - y - c_{1}s)} \gamma(x, \theta, 0)d\theta$$

$$- \frac{c_{5}}{c_{6}} e^{-c_{2}x} \delta(x - y - c_{1}s), \qquad (110)$$

$$\text{if } 0 \leq y \leq c_{4}s,$$

$$\gamma(x, s, y) = -\int_{y-c_4 s}^{y+c_1 s} \frac{kc_5}{c_6(c_1 + c_4)} e^{-\frac{c_1 c_2}{c_1 + c_4} \theta - \frac{c_2 c_4}{c_1 + c_4} (y + c_1 s)} \times \delta(x - \theta) d\theta - \frac{c_5}{c_6} e^{-c_2 x} \delta(x - y - c_1 s),$$
(111)

$$\gamma(x, s, y) = -\int_{y-c_4 s}^{\frac{c_1+c_4}{c_1}L - \frac{c_4}{c_1}y - c_4 s} \frac{kc_5}{c_6(c_1 + c_4)} \times e^{-\frac{c_1c_2}{c_1+c_4}\theta - \frac{c_2c_4}{c_1+c_4}(y+c_1s)} \delta(x-\theta)d\theta - \frac{kc_5}{c_1c_6} e^{-c_2L},$$
if $L - c_1 s < y \le L$. (112)

We use successive approximations method for (110) and get the following iterations:

$$\gamma^{n+1}(x,s,y) = -\frac{kc_5}{c_6(c_1+c_4)} e^{-\frac{c_1c_2}{c_1+c_4}x - \frac{c_2c_4}{c_1+c_4}(y+c_1s)}$$

$$-\frac{c_5}{c_6} e^{-c_2x} \delta(x-y-c_1s)$$

$$+ \int_0^{s-\frac{y}{c_4}} \frac{c_1c_5c_7}{c_1+c_4} e^{\frac{c_2c_4}{c_1+c_4}(c_1\theta-y-c_1s)} \gamma^n(x,\theta,0) d\theta,$$
if $0 \le x \le y+c_1s$, for $n = 0, 1, 2, ...$ (113)

Let

$$\Delta \gamma^n(x, s, y) = \gamma^{n+1}(x, s, y) - \gamma^n(x, s, y), \tag{114}$$

hen,

$$\Delta \gamma^{0}(x, s, y) = -\frac{kc_{5}}{c_{6}(c_{1} + c_{4})} e^{-\frac{c_{1}c_{2}}{c_{1} + c_{4}}x - \frac{c_{2}c_{4}}{c_{1} + c_{4}}(y + c_{1}s)}$$

$$-\frac{c_{5}}{c_{6}} e^{-c_{2}x} \delta(x - y - c_{1}s), \qquad (115)$$

$$\Delta \gamma^{n}(x, s, y) = \int_{\frac{x}{c_{1}}}^{s - \frac{y}{c_{4}}} \frac{c_{1}c_{5}c_{7}}{c_{1} + c_{4}} e^{\frac{c_{2}c_{4}}{c_{1} + c_{4}}(c_{1}\theta - y - c_{1}s)}$$

$$\times \Delta \gamma^{n-1}(x, \theta, 0) d\theta, \qquad (116)$$

After a series of iterations of γ^n , we have

$$\Delta \gamma^{n} = -\frac{c_{5}}{c_{6}} e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}x - \frac{c_{2}c_{4}}{c_{1}+c_{4}}(y+c_{1}s)} \times \left[\frac{c_{5}c_{7}(c_{1}c_{5}c_{7}(s-\frac{y}{c_{4}}-\frac{x}{c_{1}}))^{n-1}}{(n-1)!(c_{1}+c_{4})^{n}} - \frac{k(c_{1}c_{5}c_{7}(s-\frac{y}{c_{4}}-\frac{x}{c_{1}}))^{n}}{n!(c_{1}+c_{4})^{n+1}} \right]$$
if $0 \le y \le c_{4}s - \frac{c_{4}}{c_{1}}x$, (117)

then.

$$\gamma(x, s, y) = \sum_{n=1}^{+\infty} \Delta \gamma^n(x, s, y) + \Delta \gamma^0(x, s, y)$$

$$= -\frac{c_5(k + c_5 c_7)}{c_6(c_1 + c_4)} e^{-c_2(x+y)} - \frac{c_5}{c_6} e^{-c_2 x} \delta(x - y - c_1 s),$$
if $0 \le y \le c_4 s - \frac{c_4}{c_1} x$. (118)

After a direct computing from (111) and (112), we reach (52).

APPENDIX B THE PROOF OF LEMMA 1

First, we prove that the transformation is continuous. Substitute $x = c_1 s$ into the first and the second case of the transformation (64), which gives

$$T_{1}[\psi(t)](c_{1}s,s) + Z_{1}[z(t)](c_{1}s) + Y_{1}[v(t)](c_{1}s)$$

$$= \psi(c_{1}s,s,t)$$

$$- \int_{0}^{s} c_{1}c_{2}e^{-c_{1}c_{2}\tau}\psi(c_{1}s-c_{1}\tau,s-\tau,t)d\tau$$

$$+ \int_{0}^{s} k\psi(c_{1}s+c_{4}\tau,s-\tau,t)d\tau$$

$$- \int_{0}^{s} \int_{c_{1}s-c_{1}\tau}^{c_{1}s+c_{4}\tau} \frac{kc_{1}c_{2}}{c_{1}+c_{4}}e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(c_{1}s-y+c_{4}\tau)}$$

$$\times \psi(y,s-\tau,t)dyd\tau$$

$$+ \int_{0}^{c_{1}s+c_{4}s} \frac{kc_{5}}{c_{6}(c_{1}+c_{4})}e^{-c_{1}c_{2}s-\frac{c_{2}c_{4}}{c_{1}+c_{4}}y}z(y,t)dy$$

$$-\frac{c_{5}c_{7}}{c_{6}}e^{-c_{1}c_{2}s}v(0,t) - \frac{k}{c_{6}}v(c_{1}s+c_{4}s,t), \tag{119}$$

and

$$T_{2}[\psi(t)](c_{1}s,s) + Z_{2}[z(t)](c_{1}s) + Y_{2}[v(t)](c_{1}s)$$

$$= \psi(c_{1}s,s,t)$$

$$- \int_{0}^{s} c_{1}c_{2}e^{-c_{1}c_{2}\tau}\psi(c_{1}s-c_{1}\tau,s-\tau,t)d\tau$$

$$+ \int_{0}^{s} k\psi(c_{1}s+c_{4}\tau,s-\tau,t)d\tau$$

$$- \int_{0}^{s} \int_{c_{1}s-c_{1}\tau}^{c_{1}s+c_{4}\tau} \frac{kc_{1}c_{2}}{c_{1}+c_{4}}e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(c_{1}s-y+c_{4}\tau)}$$

$$\times \psi(y,s-\tau,t)dyd\tau$$

$$+ \frac{c_{5}}{c_{6}}e^{-c_{1}c_{2}s}z(0,t)$$

$$+ \int_{0}^{c_{1}s+c_{4}s} \frac{kc_{5}}{c_{6}(c_{1}+c_{4})}e^{-c_{1}c_{2}s-\frac{c_{2}c_{4}}{c_{1}+c_{4}}y}z(y,t)dy$$

$$- \frac{k}{c_{c}}v(c_{1}s+c_{4}s,t), \tag{120}$$

It shows that (119) equals to (120) by using the the boundary condition (26). In a similar way, one can get

$$T_{2}[\psi(t)](L - c_{4}s, s) = T_{3}[\psi(t)](L - c_{4}s, s),$$

$$Z_{2}[z(t)](L - c_{4}s) = Z_{3}[z(t)](L - c_{4}s),$$

$$Y_{2}[v(t)](L - c_{4}s, s) = \frac{k}{c_{6}}v(L, t),$$

which implies the transformation (64) in the second and third case has a same value at $x = L - c_4 s$.

Second, it is obvious that the transformation is bounded from the explicit form of each kernel function.

Third, differentiating the transformation (64) with respect to t and s, respectively, and then substituting them into (36) and (37), one can prove the system (24)-(29) can be transformed into (32)-(37) via (64) after a lengthy computation. The details are shown as follows: in the case of $0 \le x < c_1 s$,

$$\beta_{s}(x,s,t) = \psi_{s}(x,s,t) + \frac{c_{4}c_{5}(k+c_{5}c_{7})}{c_{6}(c_{1}+c_{4})} e^{-c_{2}(c_{4}s+\frac{c_{1}-c_{4}}{c_{1}}x)} z(c_{4}(s-\frac{x}{c_{1}}),t) + \frac{kc_{4}c_{5}}{c_{6}(c_{1}+c_{4})} e^{-c_{2}(x+c_{4}s)} z(x+c_{4}s,t) - \frac{kc_{4}c_{5}}{c_{6}(c_{1}+c_{4})} e^{\frac{c_{2}(c_{4}-c_{1})}{c_{1}}x-c_{2}c_{4}s} z(c_{4}(s-\frac{x}{c_{1}}),t) - \int_{c_{4}(s-\frac{x}{c_{1}})}^{x+c_{4}s} \frac{kc_{1}c_{2}c_{4}c_{5}}{c_{6}(c_{1}+c_{4})^{2}} e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}x-\frac{c_{2}c_{4}}{c_{1}+c_{4}}(y+c_{1}s)} \times z(y,t)dy - \frac{c_{4}c_{5}c_{7}}{c_{6}} e^{-c_{2}x}v_{x}(c_{4}(s-\frac{x}{c_{1}}),t) - \frac{kc_{4}}{c_{6}}v_{x}(x+c_{4}s,t) - \frac{c_{1}c_{5}c_{7}(k+c_{5}c_{7})}{c_{6}(c_{1}+c_{4})} e^{-c_{2}x}v(c_{4}(s-\frac{x}{c_{1}}),t) - \int_{0}^{\frac{x}{c_{1}}} c_{1}c_{2}e^{-c_{1}c_{2}r}\psi_{s}(x-c_{1}r,s-r,t)dr + \int_{s}^{s} c_{5}c_{7}e^{-c_{2}x}\psi(c_{4}(s-\frac{x}{c_{1}}),0,t) + \int_{0}^{s} k\psi_{s}(x+c_{4}r,s-r,t)dr + k\psi(x+c_{4}s,0,t) - \int_{0}^{s} \int_{\max\{x-c_{1}r,c_{4}(r-\frac{x}{c_{1}})\}}^{x+c_{4}r} \frac{kc_{1}c_{2}}{c_{1}+c_{4}}e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(x-y+c_{4}r)} \times \psi_{s}(y,s-r,t)dydr - \int_{c_{4}(s-\frac{x}{c_{1}})}^{x+c_{4}s} \frac{kc_{1}c_{2}}{c_{1}+c_{4}}e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(x-y+c_{4}s)}\psi(y,0,t)dydr; (121)$$

in the case of $c_1 s < x \le L - c_4 s$,

$$\beta_s(x, s, t) = \psi_s(x, s, t) - \frac{c_1 c_5}{c_6} e^{-c_2 x} z_x(x - c_1 s, t)$$

$$+ \frac{kc_{4}c_{5}}{c_{6}(c_{1} + c_{4})} e^{-c_{2}(x+c_{4}s)} z(x + c_{4}s, t)$$

$$+ \frac{kc_{1}c_{5}}{c_{6}(c_{1} + c_{4})} e^{-c_{2}x} z(x - c_{1}s, t)$$

$$- \int_{x-c_{1}s}^{x+c_{4}s} \frac{kc_{1}c_{2}c_{4}c_{5}}{c_{6}(c_{1} + c_{4})^{2}} e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}x - \frac{c_{2}c_{4}}{c_{1}+c_{4}}(y+c_{1}s)} z(y, t) dy$$

$$- \frac{kc_{4}}{c_{6}} v_{x}(x + c_{4}s, t)$$

$$- \int_{0}^{s} c_{1}c_{2}e^{-c_{1}c_{2}r} \psi_{s}(x - c_{1}r, s - r, t) dr$$

$$- c_{1}c_{2}e^{-c_{1}c_{2}s} \psi(x - c_{1}s, 0, t)$$

$$+ \int_{0}^{s} k\psi_{s}(x + c_{4}r, s - r, t) dr$$

$$+ k\psi(x + c_{4}s, 0, t)$$

$$- \int_{0}^{s} \int_{x-c_{1}r}^{x+c_{4}r} \frac{kc_{1}c_{2}}{c_{1}+c_{4}} e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(x-y+c_{4}s)}$$

$$\times \psi_{s}(y, s - r, t) dy dr$$

$$- \int_{x-c_{1}s}^{x+c_{4}s} \frac{kc_{1}c_{2}}{c_{1}+c_{4}} e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(x-y+c_{4}s)}$$

$$\times \psi(y, 0, t) dy; \qquad (122)$$

and in the case of $L - c_4 s < x \le L$,

$$\begin{split} &\beta_{s}(x,s,t) = \psi_{s}(x,s,t) \\ &- \frac{c_{1}c_{5}}{c_{6}} \mathrm{e}^{-c_{2}x} z_{x}(x-c_{1}s,t) \\ &- \frac{kc_{1}c_{5}}{c_{6}(c_{1}+c_{4})} \mathrm{e}^{-c_{2}L} z(\frac{c_{1}+c_{4}}{c_{4}}L - \frac{c_{1}}{c_{4}}x - c_{1}s,t) \\ &+ \frac{kc_{1}c_{5}}{c_{6}(c_{1}+c_{4})} \mathrm{e}^{-c_{2}x} z(x-c_{1}s,t) \\ &- \int_{x-c_{1}s}^{\frac{c_{1}+c_{4}}{c_{4}}L - \frac{c_{1}}{c_{4}}x - c_{1}s} \frac{kc_{1}c_{2}c_{4}c_{5}}{c_{6}(c_{1}+c_{4})^{2}} \\ &\times \mathrm{e}^{-\frac{c_{1}+c_{2}}{c_{1}+c_{4}}x - \frac{c_{2}c_{4}}{c_{1}+c_{4}}(y+c_{1}s)} z(y,t) dy \\ &+ \frac{kc_{5}}{c_{6}} \mathrm{e}^{c_{2}L} z(\frac{c_{1}+c_{4}}{c_{4}}L - \frac{c_{1}}{c_{4}}x - c_{1}s,t) \\ &- \int_{0}^{s} c_{1}c_{2} \mathrm{e}^{-c_{1}c_{2}r} \psi_{s}(x-c_{1}r,s-r,t) dr \\ &- c_{1}c_{2} \mathrm{e}^{-c_{1}c_{2}s} \psi(x-c_{1}s,0,t) \\ &+ \int_{0}^{\frac{L-x}{c_{4}}} k\psi_{s}(L,s-r,t) dr \\ &+ k\psi(L,0,t) dr \\ &- \int_{0}^{s} \int_{x-c_{1}r}^{\min\{x+c_{4}r,\frac{c_{1}+c_{4}}{c_{4}}L - \frac{c_{1}}{c_{4}}x-c_{1}r\}} \frac{kc_{1}c_{2}}{c_{1}+c_{4}} \\ &\times \mathrm{e}^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}}(x-y+c_{4}s) \psi_{s}(y,s-r,t) dy dr \\ &- \int_{x-c_{1}s}^{s} \int_{c_{1}+c_{4}}^{L-\frac{c_{1}}{c_{4}}x-c_{1}r} kc_{2} \mathrm{e}^{-c_{2}L}\psi_{s}(y,s-r,t) dy dr \\ &- \int_{\frac{L-x}{c_{4}}}^{s} \int_{\frac{c_{1}+c_{4}}{c_{4}}}^{L-\frac{c_{1}}{c_{4}}x-c_{1}r} kc_{2} \mathrm{e}^{-c_{2}L}\psi_{s}(y,s-r,t) dy dr \\ &- \int_{\frac{L-x}{c_{4}}}^{s} \int_{\frac{L-x}{c_{4}}}^{s} \int_{\frac{L-x}{c_{4}}}^{s} \int_{\frac{L-x}{c_{4}}}^{s} \mathrm{e}^{-c_{4}} \mathrm{e}^{-c_{4}}$$

$$-\int_{\frac{c_1+c_4}{c_4}L-\frac{c_1}{c_4}x-c_1s}^{L} kc_2 e^{-c_2L} \psi(y,0,t) dy.$$
 (123)

Then, differentiating (64) with respect of t, substituting (24), (25), (27) and (28) to the time derivative of (64), and using integration by parts, we have β_t in three cases as follows: in the case of $0 \le x < c_1 s$,

$$\begin{split} &\beta_t(x,s,t) = \psi_t(x,s,t) \\ &- \frac{c_1c_5(k+c_5c_7)}{c_6(c_1+c_4)} \mathrm{e}^{-c_2(c_4s+\frac{c_1-c_4}{c_1}x)} z(c_4(s-\frac{x}{c_1}),t) \\ &+ \frac{c_1c_5(k+c_5c_7)}{c_6(c_1+c_4)} \mathrm{e}^{-c_2x} z(0,t) \\ &- \int_0^{c_4(s-\frac{x}{c_1})} \frac{c_1c_2c_5(k+c_5c_7)}{c_6(c_1+c_4)} \mathrm{e}^{-c_2x} \psi(y,0,t) dy \\ &- \int_0^{c_4(s-\frac{x}{c_1})} \frac{c_1c_2(k+c_5c_7)}{c_1+c_4} \mathrm{e}^{-c_2x} \psi(y,0,t) dy \\ &- \frac{kc_1c_5}{c_6(c_1+c_4)} \mathrm{e}^{-c_2(x+c_4s)} z(x+c_4s,t) \\ &+ \frac{kc_1c_5}{c_6(c_1+c_4)} \mathrm{e}^{\frac{c_2(c_4-c_1)}{c_1}x-c_2c_4s} z(c_4(s-\frac{x}{c_1}),t) \\ &- \int_{c_4(s-\frac{x}{c_1})}^{x+c_4s} \frac{kc_1c_2c_4c_5}{c_6(c_1+c_4)^2} \mathrm{e}^{-\frac{c_1c_2}{c_1+c_4}x-\frac{c_2c_4}{c_1+c_4}(y+c_1s)} \\ &\times z(y,t) dy \\ &- \int_{c_4(s-\frac{x}{c_1})}^{x+c_4s} \frac{kc_1c_2}{c_1+c_4} \mathrm{e}^{-\frac{c_1c_2}{c_1+c_4}(x-y+c_4s)} \\ &\times \psi(y,0,t) dy \\ &- \frac{c_4c_5c_7}{c_6} \mathrm{e}^{-c_2x}v_x(c_4(s-\frac{x}{c_1}),t) \\ &+ \frac{c_5^2c_7}{c_6} \mathrm{e}^{-c_2(c_4s+\frac{c_1-c_4}{c_1}x)} z(c_4(s-\frac{x}{c_1}),t) \\ &+ c_5c_7\mathrm{e}^{-c_2x}\psi(c_4(s-\frac{x}{c_1}),0,t) \\ &- \frac{kc_4}{c_6}v_x(x+c_4s,t) \\ &+ k\psi(x+c_4s,0,t) \\ &- \frac{c_1c_5c_7(k+c_5c_7)}{c_6(c_1+c_4)} \mathrm{e}^{-c_2x}v(c_4(s-\frac{x}{c_1}),t) \\ &+ \frac{c_1c_5c_7(k+c_5c_7)}{c_6(c_1+c_4)} \mathrm{e}^{-c_2x}v(0,t) \\ &+ \int_0^{c_4(s-\frac{x}{c_1})} \frac{c_1c_2c_5(k+c_5c_7)}{c_6(c_1+c_4)} \mathrm{e}^{-c_2x}\psi(y,0,t) dy \\ &- \int_0^{c_1} c_1c_2\mathrm{e}^{-c_1c_2r}\psi_t(x-c_1r,s-r,t) dr \\ &+ \int_0^s k\psi_t(x+c_4r,s-r,t) dr \end{aligned}$$

$$-\int_{0}^{s} \int_{\max\{x-c_{1}r,c_{4}(r-\frac{x}{c_{1}})\}}^{x+c_{4}r} \frac{kc_{1}c_{2}}{c_{1}+c_{4}} e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(x-y+c_{4}r)}$$

$$\times \psi_{t}(y,s-r,t)dydr; \tag{124}$$

in the case of $c_1 s < x \le L - c_4 s$,

$$\beta_{t}(x,s,t) = \psi_{t}(x,s,t)$$

$$-\frac{c_{1}c_{5}}{c_{6}}e^{-c_{2}x}z_{x}(x-c_{1}s,t)$$

$$-c_{1}c_{2}e^{-c_{1}c_{2}s}\psi(x-c_{1}s,0,t)$$

$$-\frac{kc_{1}c_{5}}{c_{6}(c_{1}+c_{4})}e^{-c_{2}(x+c_{4}s)}z(x+c_{4}s,t)$$

$$+\frac{kc_{1}c_{5}}{c_{6}(c_{1}+c_{4})}e^{-c_{2}x}z(x-c_{1}s,t)$$

$$-\int_{x-c_{1}s}^{x+c_{4}s}\frac{kc_{1}c_{2}c_{4}c_{5}}{c_{6}(c_{1}+c_{4})^{2}}e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}x-\frac{c_{2}c_{4}}{c_{1}+c_{4}}(y+c_{1}s)}z(y,t)dy$$

$$-\int_{x-c_{1}s}^{x+c_{4}s}\frac{kc_{1}c_{2}}{c_{1}+c_{4}}e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(x-y+c_{4}s)}$$

$$\times \psi(y,0,t)dy$$

$$-\frac{kc_{4}}{c_{6}}v_{x}(x+c_{4}s,t)$$

$$+\frac{kc_{5}}{c_{6}}e^{-c_{2}(x+c_{4}s)}z(x+c_{4}s,t)$$

$$+k\psi(x+c_{4}s,0,t)$$

$$-\int_{0}^{s}c_{1}c_{2}e^{-c_{1}c_{2}r}\psi_{t}(x-c_{1}r,s-r,t)dr$$

$$+\int_{0}^{s}k\psi_{t}(x+c_{4}r,s-r,t)dr$$

$$-\int_{0}^{s}\int_{x-c_{1}r}^{x+c_{4}r}\frac{kc_{1}c_{2}}{c_{1}+c_{4}}e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(x-y+c_{4}r)}$$

$$\times \psi_{t}(y,s-r,t)dydr; \qquad (125)$$

in the case of $L - c_4 s < x \le L$,

$$\beta_{t}(x, s, t) = \psi_{t}(x, s, t)$$

$$-\frac{c_{1}c_{5}}{c_{6}}e^{-c_{2}x}z_{x}(x - c_{1}s, t)$$

$$-c_{1}c_{2}e^{-c_{1}c_{2}s}\psi(x - c_{1}s, 0, t)$$

$$-\frac{kc_{1}c_{5}}{c_{6}(c_{1} + c_{4})}e^{-c_{2}L}z(\frac{c_{1} + c_{4}}{c_{4}}L - \frac{c_{1}}{c_{4}}x - c_{1}s, t)$$

$$+\frac{kc_{1}c_{5}}{c_{6}(c_{1} + c_{4})}e^{-c_{2}x}z(x - c_{1}s, t)$$

$$-\int_{x-c_{1}s}^{\frac{c_{1}+c_{4}}{c_{4}}L - \frac{c_{1}}{c_{4}}x - c_{1}s}\frac{kc_{1}c_{2}c_{4}c_{5}}{c_{6}(c_{1} + c_{4})^{2}}$$

$$\times e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}x - \frac{c_{1}c_{2}c_{4}}{c_{1}+c_{4}}(y + c_{1}s)}z(y, t)dy$$

$$-\int_{x-c_{1}s}^{\frac{c_{1}+c_{4}}{c_{4}}(x - c_{1}s)}\frac{kc_{1}c_{2}}{c_{1}+c_{4}}$$

$$\times e^{-\frac{c_{1}c_{2}}{c_{1}+c_{4}}(x - y + c_{4}s)}\psi(y, 0, t)dy$$

$$-\frac{kc_{5}}{c_{6}}e^{c_{2}L}z(L, t)$$

$$+\frac{kc_{5}}{c_{6}}e^{c_{2}L}z(\frac{c_{1}+c_{4}}{c_{4}}L - \frac{c_{1}}{c_{4}}x - c_{1}s, t)$$

$$-\int_{\frac{c_1+c_4}{c_4}L-\frac{c_1}{c_4}x-c_1s}^{L} kc_2 e^{-c_2L} \psi(y,0,t) dy$$

$$+\frac{kc_5}{c_6} e^{c_2L} z(L,t)$$

$$+k \psi(L,0,t) dr$$

$$-\int_{0}^{s} c_1 c_2 e^{-c_1 c_2 r} \psi_t(x-c_1 r,s-r,t) dr$$

$$+\int_{0}^{\frac{L-x}{c_4}} k \psi_t(x+c_4 r,s-r,t) dr$$

$$+\int_{\frac{L-x}{c_4}}^{s} k \psi_t(L,s-r,t) dr$$

$$-\int_{0}^{s} \int_{x-c_1 r}^{\min\{x+c_4 r,\frac{c_1+c_4}{c_4}L-\frac{c_1}{c_4}x-c_1 r\}} \frac{kc_1 c_2}{c_1+c_4}$$

$$\times e^{-\frac{c_1 c_2}{c_1+c_4}(x-y+c_4 r)} \psi_t(y,s-r,t) dy dr$$

$$-\int_{\frac{L-x}{c_4}}^{s} \int_{\frac{c_1+c_4}{c_4}L-\frac{c_1}{c_4}x-c_1 r}^{c_1} kc_2 e^{-c_2 L} \psi_t(y,s-r,t) dy dr.$$
(126)

For each case, one can reach $\beta_t(x,s,t) - \beta_s(x,s,t) = 0$. In order to get (32), (33) and (35) of the target system, we get the relation between ψ and β at s=0 via transformation (64) as follows:

$$\psi(x,0,t) = \beta(x,0,t) - \frac{c_5}{c_6} e^{-c_2 x} z(x,t) + \frac{k}{c_6} v(x,t).$$
 (127)

Substitute (127) into (24), (25) and (27), respectively, which gives (32), (33) and (35). Hence, the transformation (64) can transform the original system (24)-(29) into the target system (32)-(37).

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