Adaptive Self Recurrent Wavelet Neural Network and Sliding Mode Controller/Observer for a Slider Crank Mechanism

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Abstract: In this paper, a novel control strategy based on an adaptive self recurrent wavelet neural network (SRWNN) and a sliding mode controller/observer for a slider crank mechanism is proposed. The aim is to reduce the tracking error of the linear displacement of this mechanism while following a specified trajectory. The controller design consists of two parts. The first one is a sliding mode control strategy and the second part is an adaptive self recurrent wavelet neural network (SRWNN) controller. This controller is trained offline first, and then the SRWNN weights are updated online by the adaptive control law. Apart from the hybrid control strategy proposed in this paper, a velocity observer is implemented to replace the use of velocity sensors. This study begins with the derivation of the dynamic equations of the slider crank mechanism followed by the derivation of the proposed observer and control strategies. Finally, to demonstrate the effectiveness of the controller/observer strategy, a numerical example is supplied to analyze the variables of the system, tracking error and the estimated variables.

Keywords: Adaptive wavelet neural networks; Sliding mode control; Sliding mode observer; Slider crank mechanism

1 Introduction

The slider crank is a mechanism with several applications in industry and other fields such as gasoline and diesel engines (Wai, 2003), where the linear displacement generated by a force is converted to a circular movement or vice versa (Haddad and Chellaboina, 2002; Lin et al, 1999). Even when in the literature some traditional

approaches are used (Komaita and Furuta, 2008; Saito et al, 2009; Wai, 2003), the main objective of this study is to improve the previously mentioned works. As it is known, wavelet neural networks are implemented in different applications as shown in Alexandridis and Zapranis (2013); Min et al (2014); Yoo et al (2007, 2005, 2006). In this paper, a novel approach is presented considering these control strategies. The main idea is to develop and design an appropriate control methodology taking into account a hybrid strategy which consists of a sliding mode controller (Azar and Serrano, 2015, 2018; Azar and Zhu, 2015; Vaidyanathan and Azar, 2015; Vaidyanathan et al., 2015) together with and adaptive self recurrent wavelet neural networks (SRWNN) (Abiyev et al, 2013; Hongyan et al, 2010, 2007; Wu et al, 2014; Yoo et al, 2007; Taha and Taha, 2018) to stabilize this mechanical system. Note that a slider crank mechanism is a closed chain mechanical system. There are many examples of observers for nonlinear systems that are fundamental for the proposed observer shown in this paper, such as, Davila et al (2005); Kalsi et al (2009); Su et al (2007). In this paper, a full order observer is presented to obtain the estimated velocities without the need of sensors (Wai, 2003; Wang et al, 2014; Xiao and Zhu, 2014). Considering the work of Hongyan et al (2010, 2007) an adaptive self recurrent wavelet neural network and sliding mode controller for the slider crank mechanism is proposed. The reason to implement a self recurrent wavelet neural network and sliding mode controller is that this control strategy has been demonstrated to be superior to other neural network controller approaches. By implementing the separation principle, the observer and controller can be designed independently (Alexandridis and Zapranis, 2013; Lin et al, 1999; Sawada and Itamiya, 2012; Yoo et al, 2005). The self recurrent wavelet neural network controller architecture consists of four layers: the input layer, a mother wavelet layer, a product layer and an output layer (Yoo et al, 2005) which makes this controller part more efficient than other neural network controller approaches.

2 Dynamic model of the slider crank mechanism

In this section, the dynamic model of the slider crank mechanism is presented. A complete dynamic model based on the Euler-Lagrange equations is derived considering all the system states that mark a complete difference from other dynamical models found in literature (Haddad and Chellaboina, 2002; Lin et al, 1999; Saito et al, 2009). In Fig. 1, the slider crank mechanism is depicted. As can be noticed, the linear motion is converted to rotational motion by applying a force $F_1(t)$. This produces a linear motion a(t) that generates a rotational movement on each joint $\theta_1(t)$,

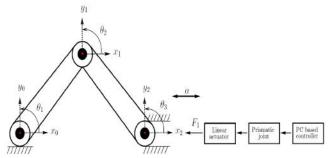


Figure 1 Slider crank mechanism

 $\theta_2(t)$ and $\theta_3(t)$. The Euler-Lagrange equations are given by:

$$D(q(t))\ddot{q}(t) + C(q(t), \dot{q}(t))\dot{q}(t) + g(q(t)) = \tau(t),$$
 (1)

where $q(t) \in \Re^n$ is the angular position and linear displacement vector, $\tau \in \Re^n$ is the torque/force vector, $D \in \Re^{n \times n}$ is the positive definite inertia matrix, $C \in \Re^{n \times n}$ is the torque vector due to Coriolis and centrifugal forces and $g \in \Re^n$ is the torque vector yielded by gravitational forces. The dimension of the slider crank mechanical system is n=4, where the vectors q(t) and $\tau(t)$ are defined by:

$$q(t) = \begin{bmatrix} \theta_1(t) \\ \theta_2(t) \\ \theta_3(t) \\ a \end{bmatrix}, \ \tau(t) = \begin{bmatrix} 0 \\ 0 \\ 0 \\ F_1(t) \end{bmatrix}$$
 (2)

In order to implement the proposed control strategy, the dynamic equation (1) is divided into the following individual parts (Spong et al, 2006):

$$\sum_{j=1}^{n} D_{kj}(q(t)) \ddot{q}_{j}(t) + \sum_{i=1}^{n} \sum_{j=1}^{n} C_{ijk}(q(t), \dot{q}(t)) \dot{q}_{i}(t) \dot{q}_{j}(t) + g_{k}(q(t)) = \tau_{k}(t),$$
(3)

where $D_{kj}(q(t))$ is the (j,k) element of the inertia matrix, $C_{ijk}(q(t),\dot{q}(t))$ is the (i,j) Christoffel symbol, $g_k(q(t))$ is the k element of the gravity vector and $\tau_k(t)$ is the k element of the input vector $\tau(t)$, where k=1,2,3,4 for the slider crank dynamic equation. The components of the inertia matrix are explained in the Appendix. For convenience, the j element of the angular/linear position vector and force/torque vector are given by $q_j(t)$ and $\tau_j(t)$ respectively.

3 Control system architecture and Observer design

The control system architecture for the slider crank mechanism is depicted in Fig. 2. This control strategy is based on simulation but the hardware and actuator dynamics can be added with a force to current converter (Tahmasebi et al, 2014). The proposed adaptive controller technique is formed by the following three components: (1) the adaptive self recurrent wavelet neural network controller; (2) sliding mode controller and (3) a full order nonlinear observer.

The implemented observer is based on different observer approaches for several kinds of mechanical systems. The observer is based on Davila et al (2005); Su et al (2007); De Wit and Fixot (1991), where the inverse dynamics of an n degrees of freedom mechanical system is implemented along with other techniques such as sliding mode to develop a successful observer. Examples of sliding mode observers for nonlinear systems can be found in Kalsi et al (2009); Xiao and Zhu (2014).

In order to design a suitable observer to estimate the state variables of the slider crank mechanism, the first step is to convert the dynamic model (1) of the slider crank mechanism to state space by selecting the following state variables:

$$x_1(t) = q(t), x_2(t) = \dot{q}(t)$$
 (4)

obtaining the state space model

$$\dot{x}_1(t) = x_2(t)
\dot{x}_2(t) = D^{-1}(x_1(t))[\tau(t) - C(x_1(t), x_2(t))x_2(t) - g(x_1(t))].$$
(5)

Therefore, the state space representation (5) of the slider crank mechanism is:

$$\dot{x}_1(t) = x_2(t)
\dot{x}_2(t) = f(t, x_1(t), x_2(t), \tau(t)).$$
(6)

Considering that all the states of the slider crank mechanism will be estimated from the measured position $x_1(t)$, the following observer is proposed to estimate all the state variables of this mechanism:

$$\dot{\hat{x}}_1(t) = \hat{x}_2(t)
\dot{\hat{x}}_2(t) = -k_0 \operatorname{sgn}(e(t)) + f(t, x_1(t), \hat{x}_2(t), \tau(t)) - k_1 e(t),
(7)$$

where $\hat{x}_1(t)$ and $\hat{x}_2(t)$ are the estimated position and velocity vectors, k_0 and k_1 are positive constants and $sgn(\cdot)$ is the sign function. The error signals e(t) and $\dot{e}(t)$ are defined by:

$$e(t) = \hat{x}_1(t) - x_1(t)$$

$$\dot{e}(t) = \hat{x}_2(t) - \dot{x}_1(t) = \hat{x}_2(t) - x_2(t).$$
(8)

These are the error signals implemented in (7) and they are used in the development of the error dynamics of the system employed in the convergence test of the estimated variables. Therefore, the system error dynamics is given by Su et al (2007); De Wit and Fixot (1991)

$$\ddot{e}(t) = \dot{\hat{x}}_2(t) - \dot{x}_2(t) = - k_0 \operatorname{sgn}(e(t)) + f(t, x_1(t), \hat{x}_2(t), \tau(t)) - k_1 e(t) - f(t, x_1(t), x_2(t), \tau(t)).$$
(9)

Now, defining the variable

$$r(t) = \dot{e}(t) + e(t) \tag{10}$$

The dynamic errors defined by the signal r(t) is:

$$\dot{r}(t) = -k_0 \operatorname{sgn}(\mathbf{e}(t)) + f(t, x_1(t), \hat{x}_2(t), \tau(t)) - k_1 e(t) - f(t, x_1(t), x_2(t), \tau(t)) + \hat{x}_2(t) - x_2(t).$$
(11)

The error equation (11) is important because the convergence of the observer explained in this section can be corroborated.

Theorem 1: Consider the observer obtained in (7). If e(t) = 0 as $t \to \infty$, then the asymptotic convergence of the state variables is assured.

Proof: This condition is tested by selecting an appropriate Lyapunov functional V(r(t), e(t)) to prove that $\dot{V}(r(t), e(t)) \leq 0$. Consider the following Lyapunov functional:

$$V(r(t), e(t)) = \frac{1}{2}r^{T}(t)r(t) + \frac{1}{2}e^{T}(t)e(t).$$
 (12)

Now, deriving (12) we obtain

$$\dot{V}(r(t), e(t)) = r^{T}(t)\dot{r}(t) + e^{T}(t)\dot{e}(t).$$
(13)

Substituting the derivatives of the error signals $\dot{r}(t)$ and $\dot{e}(t)$ from (11) and (8), respectively, yields:

$$\dot{V}(r(t), e(t)) = -r^{T}(t)k_{0} \operatorname{sgn}(e(t))
+ r^{T}(t)f(t, x_{1}(t), \hat{x}_{2}(t), \tau(t)) - r^{T}(t)k_{1}e(t)
- r^{T}(t)f(t, x_{1}(t), x_{2}(t), \tau(t)) + r^{T}(t)\hat{x}_{2}(t)
- r^{T}(t)x_{2}(t) + e^{T}(t)\hat{x}_{2}(t) - e^{T}(t)x_{2}(t).$$
(14)

Rearranging (14) we have

$$\dot{V}(r(t), e(t)) = -r^{T}(t)k_{0} \operatorname{sgn}(e(t))
- r^{T}(t)(f(t, x_{1}(t), x_{2}(t), \tau(t))
- f(t, x_{1}(t), \hat{x}_{2}(t), \tau(t))) - r^{T}(t)k_{1}e(t)
- r^{T}(t)(x_{2}(t) - \hat{x}_{2}(t))
- e^{T}(t)(x_{2}(t) - \hat{x}_{2}(t)).$$
(15)

Then, an upper bound for the derivative of the Lyapunov function is found in the following form (Liu, 1999; Xiang and Siow, 2004):

$$\dot{V}(r(t), e(t)) \leq -k_{0min} \| r^{T}(t) \operatorname{sgn}(e(t)) \|
-k_{1min} \| r^{T}(t) e(t) \|
- \| r^{T}(t) \| \| f(t, x_{1}(t), x_{2}(t), \tau(t)) - f(t, x_{1}(t), \hat{x}_{2}(t), \tau) \|
- \| r^{T}(t) \| \| x_{2}(t) - \hat{x}_{2}(t) \| - \| e^{T}(t) \| \| x_{2}(t) - \hat{x}_{2}(t) \| ,$$
(16)

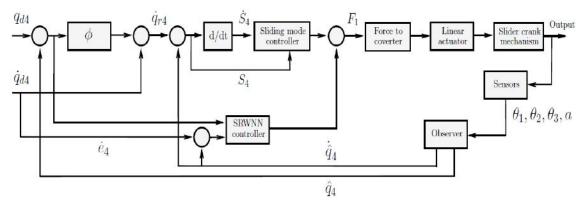


Figure 2 Control system architecture

where

$$k_{0min}, k_{1min} > 0, k_{0min} = \min_{i \in n} k_{0i}, k_{1min} = \min_{i \in n} k_{1i},$$
(17)

Thus, the global asymptotic convergence of the observer is ensured and the proof is completed.

4 Controller design

In this section, the design and development of an adaptive self recurrent wavelet neural network and sliding mode controller is described. As shown in Fig. 3, the controller architecture consists of a sliding mode and an adaptive SRWNN controller in which their input signals are determined by the error signals e(t), $\dot{e}(t)$, that are used as the inputs of the controller, and S, \dot{S} as the sliding variables.

4.1 Self recurrent wavelet neural network controller structure and offline training

The inputs of the SRWNN controller part are established by the dynamic errors of the slider crank mechanism that, in this case, are the linear displacement error $e_4(t)$ and the linear velocity error $\dot{e}_4(t)$. A schematic diagram of the self recurrent wavelet neural network controller is shown in Fig. 3. The hybrid control strategy consists of the following four layers as defined in (Alexandridis and Zapranis, 2013; Yoo et al., 2005, 2006):

Layer 1: The input layer of the SRWNN controller part with inputs $e_4(t)$ and $\dot{e}_4(t)$.

Layer 2: The mother wavelet layer in which each node is a mother wavelet with a self-feedback loop. In this article a function similar to $\phi(x(t)) = -x(t)exp(-\frac{1}{2}x^2(t))$ is implemented as a mother wavelet function.

Layer 3: The product layer, where the product of the mother wavelets is obtained.

Layer 4: The output layer, where the sum of the products obtained in layer 3 and the weighted inputs are computed.

The neural network architecture for the SRWNN controller part of the slider crank mechanism is designed

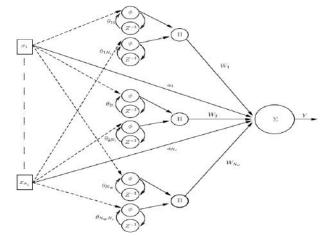


Figure 3 SRWNN controller architecture

with two input signals, the linear displacement error $e_4(t)$ and the linear velocity error $\dot{e}_4(t)$. The mother wavelets of the second layer are defined by:

$$\phi(z_{jk}(n)) = -\left(\frac{u_{jk}(n) - m_{jk}}{d_{jk}}\right) exp\left(-\frac{1}{2}\left(\frac{u_{jk}(n) - m_{jk}}{d_{jk}}\right)^2\right),$$
(18)

where

$$\phi(z_{jk}(n)) = \phi\left(\frac{u_{jk}(n) - m_{jk}}{d_{jk}}\right),\tag{19}$$

with

$$z_{jk}(n) = \frac{u_{jk}(n) - m_{jk}}{d_{jk}},\tag{20}$$

where m_{jk} and d_{jk} are the translation and dilation factors of the mother wavelet (Yoo et al, 2005, 2006). The subscript j, k is the k input term of the j wavelet. The inputs of this layer, in discrete time n, are defined by:

$$u_{ik}(n) = x_k(n) + \phi_{ik}(n-1)\theta_{ik},$$
 (21)

where θ_{jk} is the weight of the auto-feedback loop shown in Fig. 3 due to the feedback in each mother wavelet in layer 2. In layer 3, the product of each mother wavelet or wavelon is denoted by:

$$\Phi_{j}(x(n)) = \prod_{k=1}^{N_{i}} \phi(z_{jk}(n)) =$$
 and the partial derivative by:
$$\prod_{k=1}^{N_{i}} \left[-\left(\frac{u_{jk}(n) - m_{jk}}{d_{jk}}\right) exp\left(-\frac{1}{2}\left(\frac{u_{jk}(n) - m_{jk}}{d_{jk}}\right)^{2}\right) \right] \cdot \frac{\partial J(n)}{\partial W(n)} = -e(n)\frac{\partial y(n)}{\partial W(n)}.$$

$$(22)$$

The signal provided by the output layer is given by:

$$y(n) = \sum_{j=1}^{N_w} w_j \Phi_j(x(n)) + \sum_{k=1}^{N_i} a_k x_k(n),$$
(23)

where w_j is the connection weight between the product node and the output node, and a_k is the connection weight between the input node and the output node. In order to compute the weights of the adaptive SRWNN controller for initialization purpose, a weight vector Wis defined as:

$$W = [a_k, m_{jk}, d_{jk}, \theta_{jk}, w_j]^T,$$
(24)

where the weights explained in this subsection are grouped into one vector in order to train the adaptive SRWNN controller part offline before the real time implementation.

4.2 Offline Training

The offline training of the SRWNN controller part of this strategy is crucial for a suitable initialization of the adaptive SRWNN weight vector. The training method implemented for this purpose is the well-known gradient descent method (Alexandridis and Zapranis, 2013; Yoo et al, 2007, 2005, 2006) which consists of the minimization of an objective function as described below.

$$J(n) = \frac{1}{2} [y_d(n) - y(n)]^2 = \frac{1}{2} e^2(n),$$
(25)

where $y_d(n)$ is the desired output and $y(n) = \hat{y}_d(n)$ is the actual output of the adaptive SRWNN controller part. In this method the weight vector W is updated in each iteration while minimizing the functional cost (25). It is defined in (Yoo et al., 2007) as:

$$W(n+1) = W(n) + \Delta W(n) = W(n) + \bar{\eta} \left(-\frac{\partial J(n)}{\partial W(n)} \right),$$
(26)

where

$$\bar{\eta} = diag[\eta^a, \eta^m, \eta^d, \eta^\theta, \eta^w], \tag{27}$$

and the partial derivative of the functional cost is given by:

$$\frac{\partial J(n)}{\partial W(n)} = -e(n)\frac{\partial y(n)}{\partial W(n)}.$$
(28)

Applying the chain rule recursively, the adjustments of the weights of each layer can be obtained. The components of the weight vector are given by:

$$\frac{\partial y(n)}{\partial a_k(n)} = x_k(n),\tag{29}$$

$$\frac{\partial y(n)}{\partial m_{jk}(n)} = -\frac{w_j}{d_{jk}} \frac{\partial \Phi_j(x(n))}{\partial z_{jk}(n)},$$
(30)

$$\frac{\partial y(n)}{\partial d_{jk}(n)} = -\frac{w_j}{d_{jk}} z_{jk}(n) \frac{\partial \Phi_j(x(n))}{\partial z_{jk}(n)},$$
(31)

$$\frac{\partial y(n)}{\partial \theta_{jk}(n)} = \frac{w_j}{d_{jk}} \phi_{jk}(n-1) \frac{\partial \Phi_j(x(n))}{\partial z_{jk}(n)},$$
(32)

$$\frac{\partial y(n)}{\partial w_j(n)} = \Phi_j(x(n)),\tag{33}$$

where

$$\frac{\partial \Phi_j(x(n))}{\partial z_{jk}(n)} = (z_{jk}^2(n) - 1) exp\left(-\left(\frac{1}{2}\right) z_{jk}^2(n)\right). \tag{34}$$

With these partial derivatives the SRWNN controller can be trained by this iterative algorithm to initialize the weight vector of the adaptive SRWNN control.

4.3 Adaptive SRWNN and Sliding Mode Controller Design

In order to derive the necessary equations with their respective stability conditions for the control of the linear displacement of the slider crank mechanism, consider the following hybrid control law (Hongyan et al, 2007):

$$\tau_4(t) = F_1(t) = U_{smc}(t) + U_{NN}(t), \tag{35}$$

where

$$U_{smc}(t) = k_a sgn(\dot{s}_4(t)) + k_b s_4(t)$$
(36)

is the sliding mode controller part of the overall controller and $k_a, k_b > 0$ are the sliding gains, \dot{s}_4 and s_4 the derivative and original variables for the dynamic errors of the slider crank mechanism defined later. Moreover,

$$U_{NN}(t) = \sum_{j=1}^{N_w} w_j \Phi_j(x(n)) + \sum_{k=1}^{N_i} a_k x_k(n)$$
 (37)

is the adaptive self recurrent wavelet neural network controller part of the overall controller where its weights have been defined previously with only one SRWNN controller output. To design the controller, it is necessary to include the Coriolis matrix components shown in (3).

$$C_{k,j}(q(t), \dot{q}(t)) = \sum_{i=1}^{4} C_{ijk}(q(t))\dot{q}_i(t),$$
(38)

where j, k = 1, 2, 3, 4. Define the terms of the position vector (2) as $q_j(t)$, where $q_1(t) = \theta_1(t)$, $q_2(t) = \theta_2(t)$, $q_3(t) = \theta_3(t)$ and $q_4(t) = a(t)$ are the respective angular positions and linear displacements of the slider crank mechanism, considering that there is only one input (the force $\tau_4(t) = F_1(t)$). The following error signals:

$$e_j(t) = q_{dj}(t) - q_j(t)$$
 (39)

are necessary for the development of the proposed hybrid control strategy and to test the stability of the system, where $q_{dj}(t)$ is the desired reference trajectory for the j joint (Liu, 1999; Slotine and Weiping, 1988; Xiang and Siow, 2004).

$$\dot{q}_{rj}(t) = \dot{q}_{dj}(t) + \Phi e_j(t), \tag{40}$$

$$s_j(t) = \dot{q}_{rj}(t) - \dot{q}_j(t),$$
(41)

and

$$\dot{s}_j(t) = \ddot{q}_{rj}(t) - \ddot{q}_j(t). \tag{42}$$

Due to the convergence of the dynamic errors of the observer, the following assumptions are implemented:

$$e_{j}(t) \to 0 , \dot{e}_{j}(t) \to 0 , \ddot{e}_{j}(t) \to 0$$

$$q_{j}(t) \simeq \hat{q}_{j}(t) , \dot{q}_{j}(t) \simeq \dot{\hat{q}}_{j}(t) , \ddot{q}_{j}(t) \simeq \ddot{\hat{q}}_{j}(t).$$

$$(43)$$

With these considerations, the stability of the overall system (controller, observer and slider crank mechanism) is assured by the following theorem.

Theorem 2: The proposed control law (35) stabilizes the linear displacement position of the slider crank mechanism.

Proof: This condition is tested by selecting an appropriate Lyapunov functional V(s) and proving that $\dot{V}(s) \leq 0$ assuring the global asymptotic stability of the overall control system, so $q_4(t)$ is stabilized when $t \to \infty$. In order to prove the theorem, it is necessary to define the dynamic errors of the system as follows (Seraji, 1989):

$$D_{44}(q(t))\dot{s}_{4}(t) + \sum_{j=1}^{n} D_{4j}(q(t))\dot{s}_{j}(t) + \sum_{j=1}^{n} C_{4j}(q(t), \dot{q}(t))s_{j}(t) =$$

$$-F_{1}(t) + D_{44}(q(t))(\ddot{q}_{d4}(t) + \Phi \dot{e}_{4}(t)) + \sum_{j=1}^{n} D_{4j}(q(t))(\ddot{q}_{dj}(t) + \Phi \dot{e}_{j}(t)) + \sum_{j=1}^{n} C_{4j}(q(t), \dot{q}(t))(\dot{q}_{dj}(t) + \Phi e_{j}(t)) + g_{4}(q(t)),$$

$$+\sum_{j=1}^{n} C_{4j}(q(t), \dot{q}(t))(\dot{q}_{dj}(t) + \Phi e_{j}(t)) + g_{4}(q(t)),$$

$$(44)$$

for n = 4, where D_{ij} is the (i, j) element of the inertia matrix, C_{ij} is the (i, j) element of the Coriolis matrix and $g_4(q(t))$ is the fourth element of the gravity vector. Considering that $D_{41}(q(t)) = D_{42}(q(t)) = D_{43}(q(t)) = 0$ (see Appendix), (44) becomes

$$D_{44}(q(t))\dot{s}_{4}(t) + \sum_{j=1}^{n} C_{4j}(q(t), \dot{q}(t))s_{j}(t) =$$

$$-F_{1}(t) + D_{44}(q(t))(\ddot{q}_{d4}(t) + \Phi \dot{e}_{4}(t))$$

$$+ \sum_{j=1}^{n} C_{4j}(q(t), \dot{q}(t))(\dot{q}_{dj}(t) + \Phi e_{j}(t)) + g_{4}(q(t)).$$
(45)

Then, rearranging the terms in (45) and substituting (40) and its derivative, we obtain:

$$D_{44}(q(t))\dot{s}_{4}(t) = -F_{1}(t) - \sum_{j=1}^{n} C_{4j}(q(t), \dot{q}(t))s_{j}(t) + D_{44}(q(t))\ddot{q}_{r4}(t) + \sum_{j=1}^{n} C_{4j}(q(t), \dot{q}(t))\dot{q}_{rj}(t) + g_{4}(q(t)).$$

$$(46)$$

Now, considering the linear parameterization of the mechanical system as shown in (Slotine and Weiping, 1988), the following equation is obtained:

$$D_{44}(q(t))\ddot{q}_{r4}(t) + \sum_{j=1}^{n} C_{4j}(q(t), \dot{q}(t))\dot{q}_{rj}(t) + q_{4}(q(t))$$

$$= Y_{4}(q(t), \dot{q}(t), \dot{q}_{rj}(t), \ddot{q}_{rj}(t))\widetilde{W}, \qquad (47)$$

where Y_4 is the fourth row of the dynamic parameters of the system and \widetilde{W} is the parameter estimation error. By implementing (47) in (46) and rearranging the terms, yields

$$\dot{s}_{4}(t) = -D_{44}^{-1}(q(t))F_{1}(t)$$

$$-D_{44}^{-1}(q(t))\sum_{j=1}^{n}C_{4j}(q(t),\dot{q}(t))s_{j}(t)$$

$$+D_{44}^{-1}(q(t))Y_{4}(q(t),\dot{q}(t),\dot{q}_{rj}(t),\ddot{q}_{rj}(t))\widetilde{W}. \tag{48}$$

Now, the following Lyapunov function to test the stability of the overall system is considered:

$$V(s(t)) = \frac{1}{2}s_4^2(t) + \frac{1}{2}\widetilde{W}^T\widetilde{W}. \tag{49}$$

Deriving (49) yields

$$\dot{V}(s(t)) = s_4(t)\dot{s}_4(t) + \widetilde{\widetilde{W}}^T \widetilde{W}. \tag{50}$$

Substituting (48) into (50), we obtain

$$\dot{V}(s(t)) = -s_4(t) D_{44}^{-1}(q(t)) F_1(t)
- s_4(t) D_{44}^{-1}(q(t)) \sum_{j=1}^n C_{4j}(q(t), \dot{q}(t)) s_j(t)
+ s_4(t) D_{44}^{-1}(q(t)) Y_4(q(t), \dot{q}(t), \dot{q}_{rj}(t), \ddot{q}_{rj}(t)) \widetilde{W}
+ \dot{\widetilde{W}}^T \widetilde{W}.$$
(51)

The adaptive laws for the SRWNN controller part of the overall controller is:

$$\dot{\widetilde{W}}^{T} = -s_4(t)D_{44}^{-1}(q(t))Y_4(q(t),\dot{q}(t),\dot{q}_{rj}(t),\ddot{q}_{rj}(t))\widetilde{W}.$$
(52)

Then,

$$\dot{V}(s(t)) = -s_4(t)D_{44}^{-1}(q(t))F_1(t) - s_4(t)D_{44}^{-1}(q(t))\sum_{j=1}^n C_{4j}(q(t),\dot{q}(t))s_j(t).$$
(53)

The upper limit of the Lyapunov functional derivative is:

$$\dot{V}(s(t)) \leq - \|s_4(t)D_{44}^{-1}(q(t))F_1(t)\|
- \sum_{j=1}^n \|s_4(t)D_{44}^{-1}(q(t))C_{4j}(q(t),\dot{q}(t))s_j(t)\|,
(54)$$

due to

$$\Phi \leq \Lambda$$

$$\Phi = \left\| s_4(t) D_{44}^{-1}(q(t)) \sum_{j=1}^n C_{4j}(q(t), \dot{q}(t)) s_j(t) \right\|$$

$$\Lambda = \sum_{j=1}^n \left\| s_4(t) D_{44}^{-1}(q(t)) C_{4j}(q(t), \dot{q}(t)) s_j(t) \right\|.$$
(55)

With the derivative of the Lyapunov function as shown in (54), the global asymptotic stability of the system is assured and the proposed hybrid controller strategy is suitable for the control of the slider crank mechanism.

5 Numerical Simulations

The simulation was performed in $MATLAB^{\circledR}$ 2018b and $SIMULINK^{\circledR}$ using an ODE45 solver. The objective of this simulation is to test the proposed controller/observer strategy implemented in the studied mechanism for position tracking of q_4 (estimating the velocities) and to corroborate that the tracking error is smaller than the obtained by other control strategies. The desired trajectory for the linear displacement q_4 is $q_{d4} = 0.3sin(2.5133t) + 0.3$ with the slider crank parameters shown in Table 1. A self recurrent wavelet neural network controller and a variation of the computed torque algorithm as shown in Sawada and Itamiya (2012); Wai (2003) are simulated for comparison purposes.

The parameters of the full order observer are:

$$k_0 = 1 \times 10^{-5} \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}, \tag{56}$$

Table 1 Slider crank mechanism parameters

Link	Length	Center of mass	
	(m)	(m)	$(Kg.m^2)$
1	0.3	0.15	0.18
2	0.5	0.25	0.18
3	0.10	0.05	0.18

and

$$k_1 = 1 \times 10^{-5} \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}, \tag{57}$$

while the sliding mode controller gains are $k_a = 0.01$ and $k_b = 0.01$. The neural network architecture of the adaptive SRWNN consists of two inputs $e_4(t)$ and $\dot{e}_4(t)$ with four mother wavelets or wavelons in layer 2. The initial weights calculated by the gradient descent method are:

$$a_k = [0.1999, 3.5000],$$

$$m_{jk} = [3.7000, 1.3000, 7.1000, 6.0000],$$

$$d_{jk} = [0.7000, 1.8999, 0.1500, 6.4000],$$

$$\theta_{jk} = [12.7000, 10.4000, 5.5000, 0.9000],$$

$$w_j = [2.3000, 1.2000].$$
(58)

In Fig. 4 the linear displacement error $q_{d4}(t) - q_4(t)$ is shown for the proposed and other control strategies. We can observe that the proposed hybrid controller has the smallest tracking error in comparison with (Sawada and Itamiya, 2012; Wai, 2003) considering that the accuracy is improved and the inverse dynamic is estimated by the SRWNN controller.

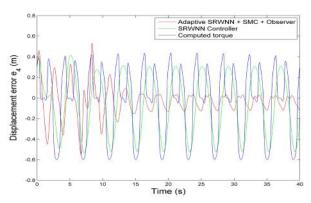


Figure 4 Tracking error for the linear displacement $q_4(t)$

In Fig. 5, the linear displacement $q_4(t)$ is depicted for the three control strategies, proving that the hybrid SRWNN control technique improves the position accuracy in comparison with (Sawada and Itamiya, 2012; Wai, 2003).

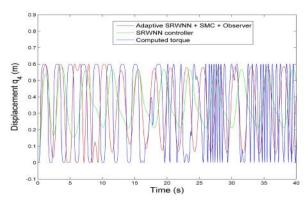


Figure 5 Linear displacement $q_4(t)$

In Fig. 6 the input forces $F_1(t)$ for the proposed and other control approaches are shown, which demonstrates that the smallest control effort is obtained with the proposed control technique in comparison with the other two strategies, avoiding the actuator saturation.

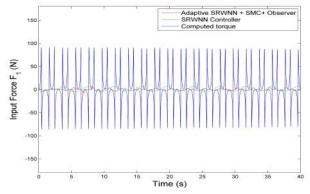


Figure 6 Input force $F_1(t)$

Finally, in Fig. 7 the angular displacement of the rotational arm $\theta_1(t)$ is shown, where a more accurate result is obtained by the proposed control strategy in comparison with Wai (2003) and a less noisy outcome is obtained in comparison with Sawada and Itamiya (2012).

6 Conclusion

The design, development and implementation of an adaptive self recurrent wavelet neural network and sliding mode controller/observer for a slider crank mechanism is presented. It is proved that a full order observer for this mechanical system can be

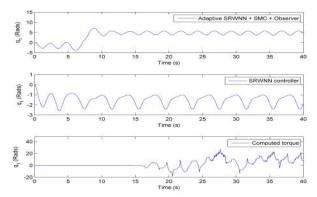


Figure 7 Angular displacement of the rotational arm $q_1(t)$ ($\theta_1(t)$)

designed taking into account the linear displacement measurement. Then, an adaptive self recurrent wavelet neural network and sliding mode controller provide a flexible approach for the control of the slider crank mechanism. Finally, it is corroborated that this controller/observer strategy yields significantly better results in comparison with other control strategies, and more important, this control approach can be used in other applications such as robotics and mechanical systems.

7 Appendix: Slider crank dynamic equations components

In this section, the symmetric inertia matrix components are shown (Spong et al, 2006). The components of the inertia tensors are represented by Ixx_i, Iyy_i, Izz_i , where i = 1, 2, 3. The masses of the rotational arm, connecting arm and linear displacement block are represented by m_1 , m_2 and m_3 respectively. In this section only the fourth row terms (59)-(60) are shown, with the diagonal terms (61),(63) and (65), due to space limitations.

$$D_{41}(q(t)) = D_{42}(q(t)) = D_{43}(q(t)) = 0,$$
(59)

$$D_{44}(q(t)) = m_1 + m_2 + m_3, (60)$$

$$D_{11}(q(t)) = (m_1 + m_2 + m_3)\gamma(t) + (m_1 + m_2 + m_3)\eta(t) + Izz_1 + Izz_2,$$
(61)

$$\gamma(t) = \left[-\sin\left(\delta\right)a - l_2\sin\left(\epsilon\right) - l_{c1}\sin\left(\theta_1(t)\right)\right]^2$$

$$\eta(t) = \left[\cos\left(\delta\right)a + l_2\cos\left(\epsilon\right) + l_{c1}\cos\left(\theta_1(t)\right)\right]^2,$$

$$D_{22}(q(t)) = (m_1 + m_2 + m_3)\rho(t) + (m_1 + m_2 + m_3)\kappa(t) + Izz_1 + Izz_2,$$
(63)

where

$$\rho(t) = \left[-\sin(\delta) \, a - l_{c2} \sin(\epsilon) \right]^2$$

$$\kappa(t) = \left[\cos(\delta) \, a + l_{c2} \cos(\epsilon) \right]^2$$

$$\delta = \theta_1(t) + \theta_2(t) + \theta_3(t)$$

$$\epsilon = \theta_1(t) + \theta_2(t)$$
(64)

$$D_{33}(q(t)) = (m_1 + m_2 + m_3)\lambda(t) + (m_1 + m_2 + m_3)\mu(t) + Izz_1 + Izz_2,$$
(65)

where

$$\lambda(t) = \left[\sin(\theta_1(t) + \theta_2(t) + \theta_3(t))\right]^2 a^2, \mu(t) = \left[\cos(\theta_1(t) + \theta_2(t) + \theta_3(t))\right]^2 a^2.$$
(66)

The lengths for each arm are given by l_1 , l_2 and l_3 . The center of masses of each link are l_{c1} , l_{c2} and l_{c3} ; g is the gravity constant and the respective angular and linear velocities are given by $\theta_i(t)$ and a, respectively.

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(62)

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Note

¹http://www.kakadusoftware.com